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SIMULATION OF PROPULSION PLANT DYNAMICS  
AND THEIR EFFECT ON SPEED CONTROL

Van Tran-Van

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# NAVAL POSTGRADUATE SCHOOL

## Monterey, California



# THESIS

SIMULATION OF PROPULSION PLANT DYNAMICS  
AND THEIR EFFECT ON SPEED CONTROL

by

Tran-Van VAN

June 1974

Thesis Advisor:

George J. Thaler

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Simulation of Propulsion Plant Dynamics

And Their Effect On Speed Control

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MASTER OF SCIENCE IN ELECTRICAL ENGINEERING

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## ABSTRACT

The dynamics of a ship propulsion plant are modeled in an all digital simulation. This model is combined with that of a mariner hull. The behavior of the speed governor is studied, and an external feedback loop is added to provide direct control of ship speed.



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# TABLE OF SYMBOLS

$X, Y, Z$	: System of reference axes fixed in the ship
$X_0, Y_0, Z_0$	: System of reference axes fixed in the ship
$m$	: Mass of ship
$\phi$	: Roll angle
$\theta$	: Pitch angle
$\psi$	: Yaw angle
$\delta$	: Rudder deflection
$K, M, N$	: Components of resultant total moment acting on a ship about the $X, Y, Z$ axes.
$p, q, r$	: Components of resultant angular velocity of the ship about the $X, Y, Z$ axes.
$I$	: Mass moments of inertia of a ship.
$Y_v$	: Partial derivative of $Y$ with respect to $v$
$Y_{\dot{v}}$	: Partial derivative of $Y$ with respect to $\dot{v}$
$Y_r$	: Partial derivative of $Y$ with respect to $r$
$Y_{\dot{r}}$	: Partial derivative of $Y$ with respect to $\dot{r}$
$N_v$	: Partial derivative of $N$ with respect to $v$
$N_{\dot{v}}$	: Partial derivative of $N$ with respect to $\dot{v}$
$N_r$	: Partial derivative of $N$ with respect to $r$
$N_{\dot{r}}$	: Partial derivative of $N$ with respect to $\dot{r}$
$n$	: Propeller angular speed
$Q_e$	: Shafting torque
$Q_p$	: Propeller reaction torque
$K_g$	: Reduction gear ratio
$W_f$	: Fuel flow rate



W : Wake fraction  
Ct : Thrust coefficient  
Cq : Torque coefficient  
T : Thrust  
Q : Torque  
p : Water mass density  
D : Propeller diameter  
 $\sigma$  : Second modified advance coefficient  
V : Ship speed





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## I. INTRODUCTION

The propulsion system of a ship is a complex, nonlinear system with several internal feedback loops. The performance of the ship itself is dependent, of course, on the capabilities and limitations of the propulsion plant, which may limit overall ship performance so that the capabilities of the hull design are not realized.

The first objective of this thesis is to model, in detail, a typical ship propulsion system, and to simulate it in the digital computer.<sup>1</sup> After validation of the computer model a simulation of the Mariner hull is to be coupled to the propulsion plant so that control system studies may be undertaken. As part of the validation procedure, behavior of the shaft speed was investigated with and without the governor.

The second objective was to use the computer model as part of a ship feedback control system. For this purpose it was decided to explore the use of a feedback loop to maintain constant ship speed in turns.

---

<sup>1</sup>IBM 360/67 DSL 360 LANGUAGE at W. R. Church Computer Center, Naval Postgraduate School.



## II. PROPULSION PLANT DYNAMICS

### A. SHIP PROPULSION EQUATIONS

The propeller angular speed is denoted by  $n$ .

The basic equation describing the angular acceleration of a propulsion shafting system is:

$$\Sigma Q = 2\pi I dn/dt$$

Where:  $\Sigma Q$  = Summation of torque =  $Q_e - Q_p$

$I$  = Polar mass moment of inertia

$n$  = Propeller angular speed

$Q_e$  = Shafting torque

$Q_p$  = Propeller reaction torque

then 
$$dn/dt = \frac{1}{2\pi I} \Sigma Q$$

For the typical gas turbine, the experimental data on the power turbine torque  $Q_e$  versus the power turbine speed is shown in Fig. A1. The torque map of Fig. A1 allows the engine torque to be determined if the engine speed and fuel flow rate  $W_f$  are known. However, this torque representation is correct only for steady state conditions.

Propeller speed  $n$ (rps) is related to the engine speed  $N_3$ (rpm) through the reduction gear ratio  $K_g$

$$N_3 = 60K_g.n$$

The block diagram for the ship propulsion system is given in Fig. A2.

The power turbine speed can be controlled by a governor acting on the gas flow. The changes in fuel flow and speed that occur can be



assumed small enough to permit the use of constant coefficients in the dynamic equations.

Using the time lag of the governor,  $\tau$ , the block diagram for the Governor is shown in Fig. A3,

Where:  $n^*$  = Rotational speed order  
 $n$  = Rotational speed  
 $ner$  =  $n^* - n$  = Speed error signal  
 $K_f$  = Fuel flow rate/RPM  
 $\tau$  = Time delay  
 $W_f$  = Fuel flow rate  
 $GOVERNOR = K_f / (1 + \tau S)$

Then the propulsion plant is shown in Fig. A4.





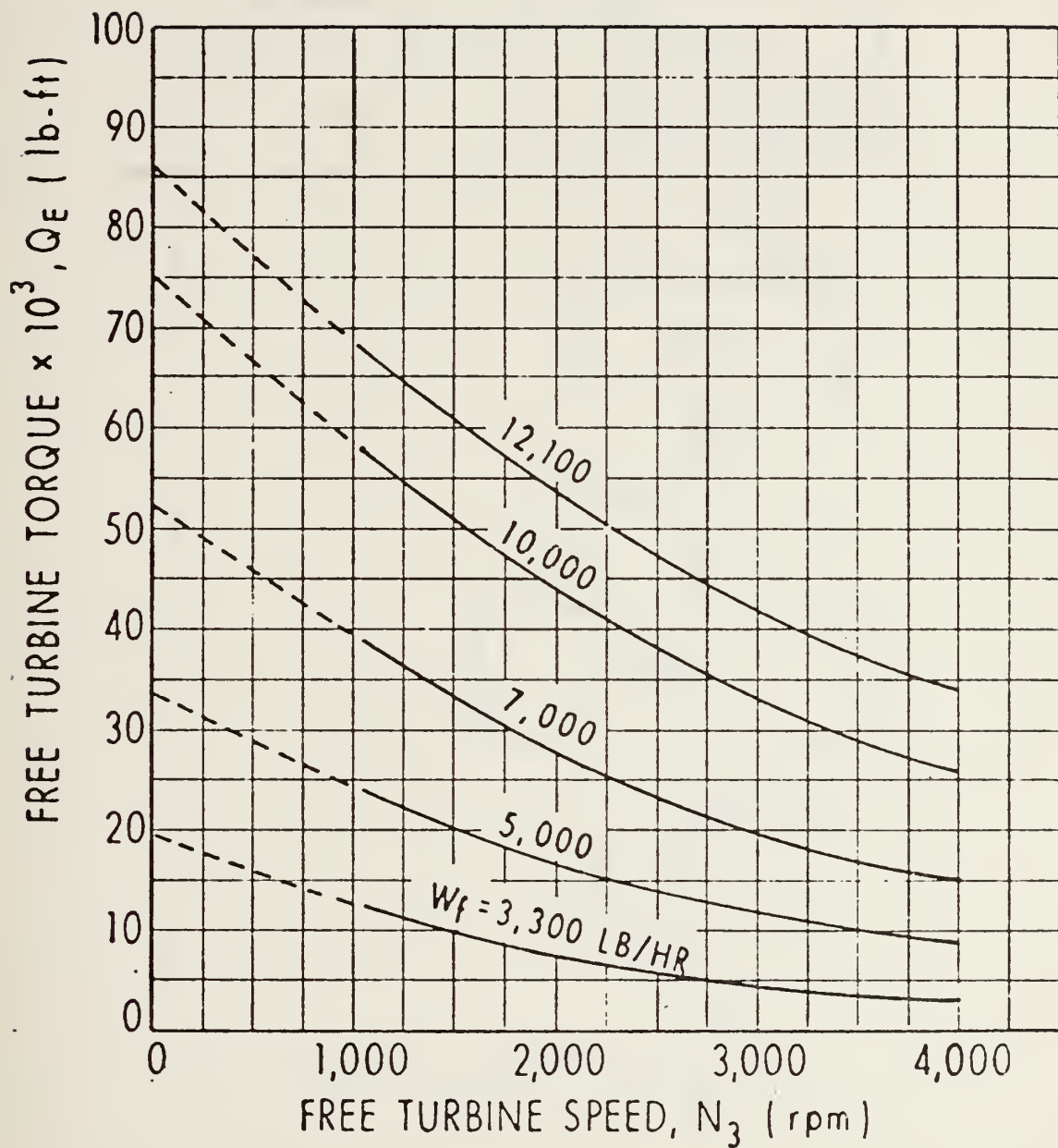


Figure A1. Engine Torque versus Engine Speed  
and Fuel Flow Rate



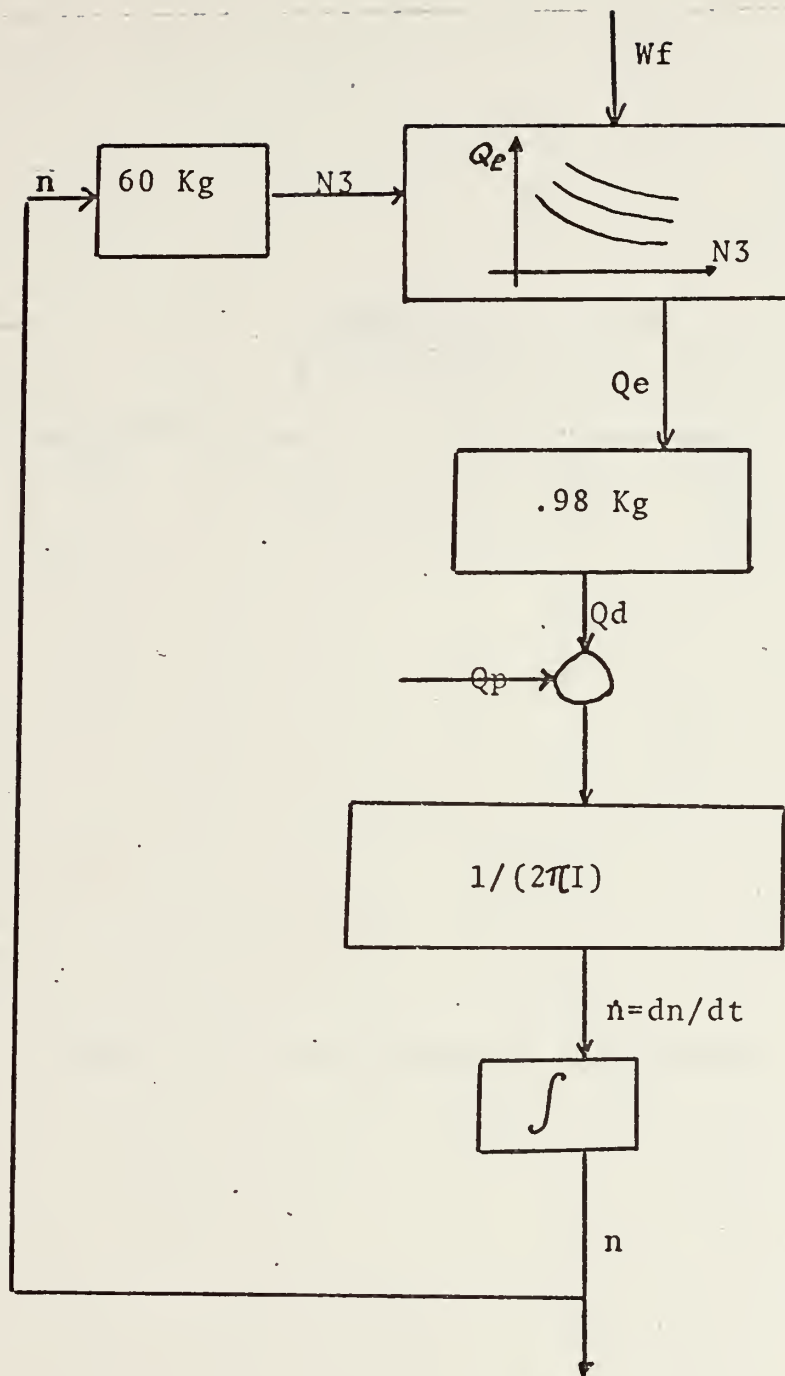


Figure A2. Block Diagram of Ship Propulsion System



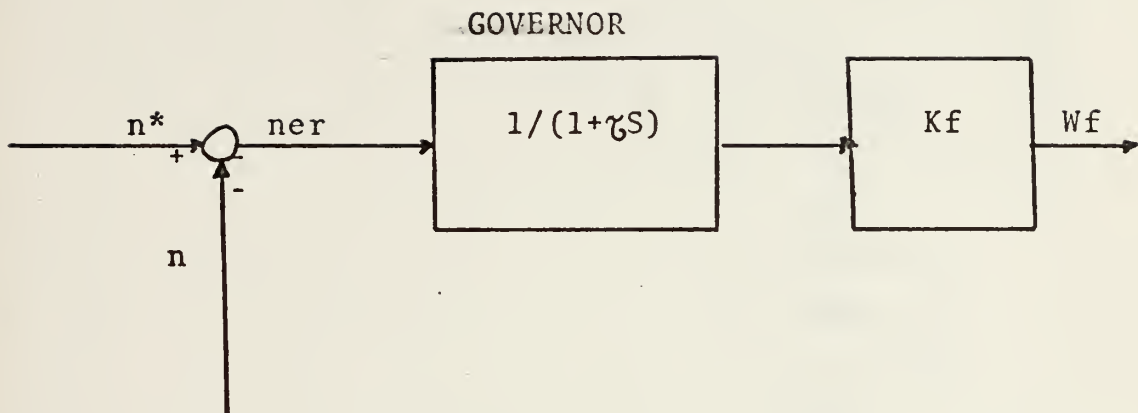


Figure A3. Block Diagram for the Governor.



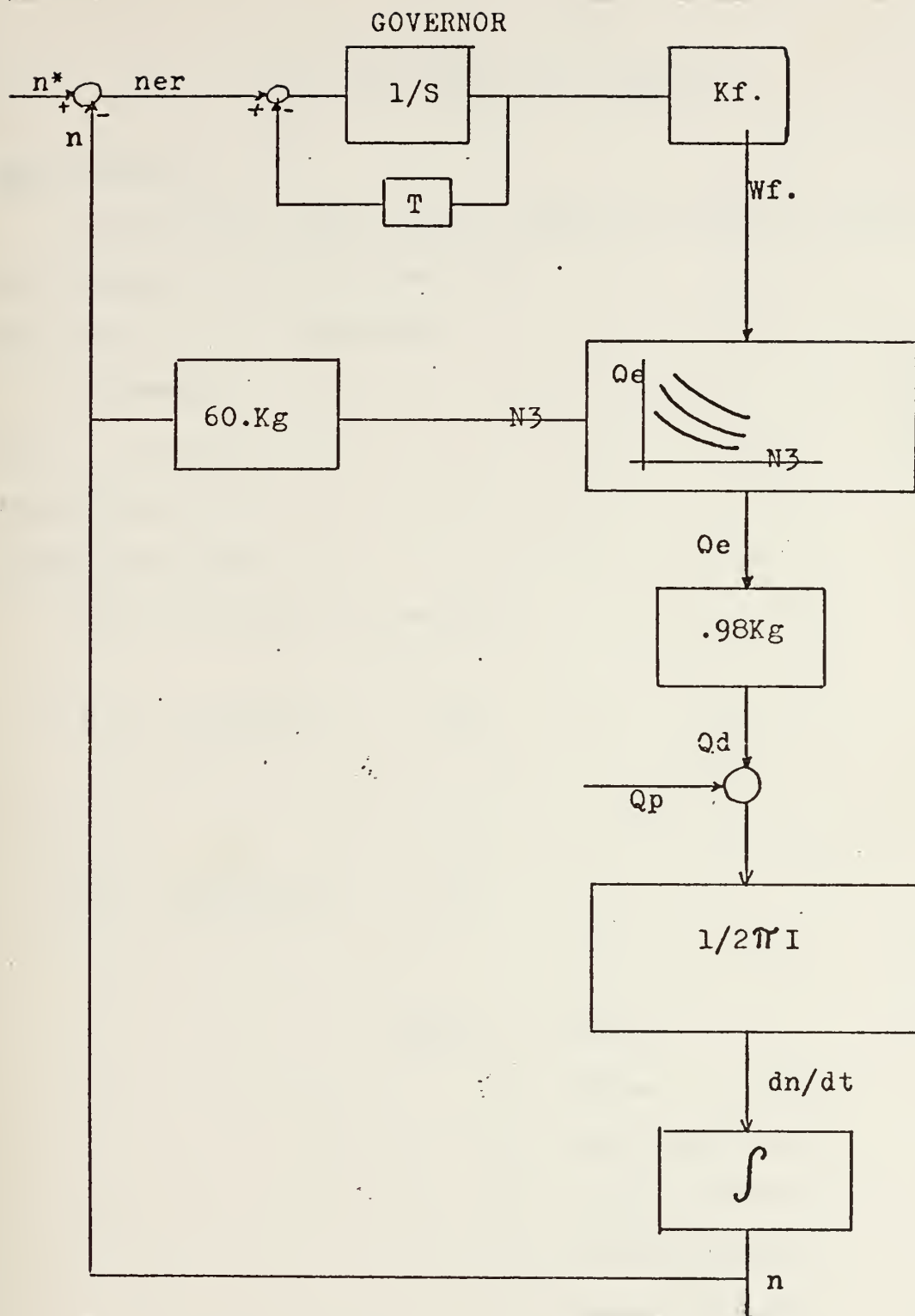


Figure A4. The Propulsion Plant with Governor





### III. HULL EQUATION AND SIMULATION

#### WAKE FRACTION: W

The difference between the ship speed and the speed of advance of the propeller  $V_p$  is called Wake speed.

The Wake fraction is defined as

$$W = (V - V_p) / V$$

thus  $V_p = V(1 - W)$

#### THRUST COEFFICIENT: $C_t$

#### TORQUE COEFFICIENT: $C_q$

For the propeller in open water

$$C_t = \frac{T}{\rho D^2 (V_p^2 + n^2 D^2)} \quad (1)$$

$$C_q = \frac{Q}{\rho D^3 (V_p^2 + n^2 D^2)} \quad (2)$$

Where  $T$  = Thrust

$Q$  = Torque

$\rho$  = Water mass density

$C_t$  = Thrust coefficient

$C_q$  = Torque coefficient

$D$  = Propeller diameter

From (1) and (2)

$$T = C_t \cdot \rho D^2 (V_p^2 + n^2 D^2)$$

$$Q = C_q \cdot \rho D^3 (V_p^2 + n^2 D^2)$$



The experimental data on the Wake fraction versus the speed of the ship is shown in Fig. B1.

The Thrust coefficient  $C_t$  and Torque coefficient  $C_q$  versus the Second modified advance coefficient are shown in Fig. B2 and Fig. B3.

Where  $V_p = V(1-W)$

And  $\sigma = (n \cdot D) / (V_p^2 + n^2 D^2)$

Total ship Resistance,  $R_t$ , is a computer look-up table giving  $R_t$  versus Ship speed  $V$  (fig. B4).



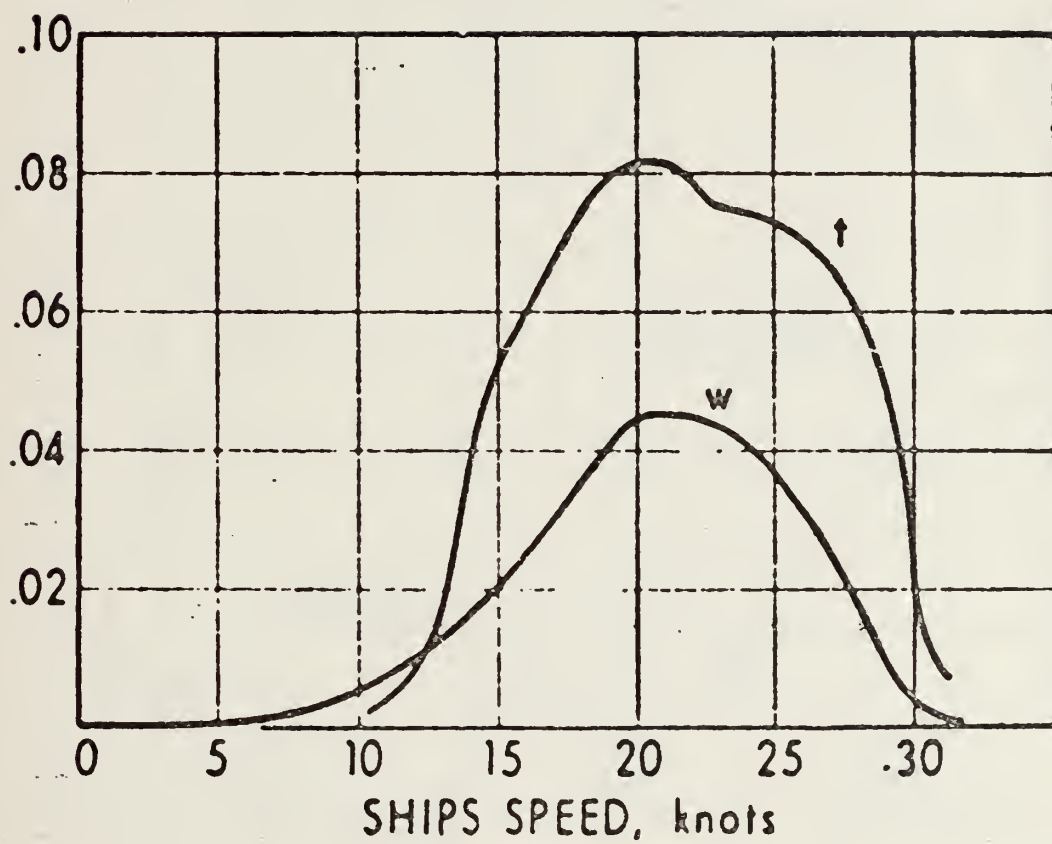


Figure B1. Wake Fraction and Thrust Deduction  
versus Ship Speed



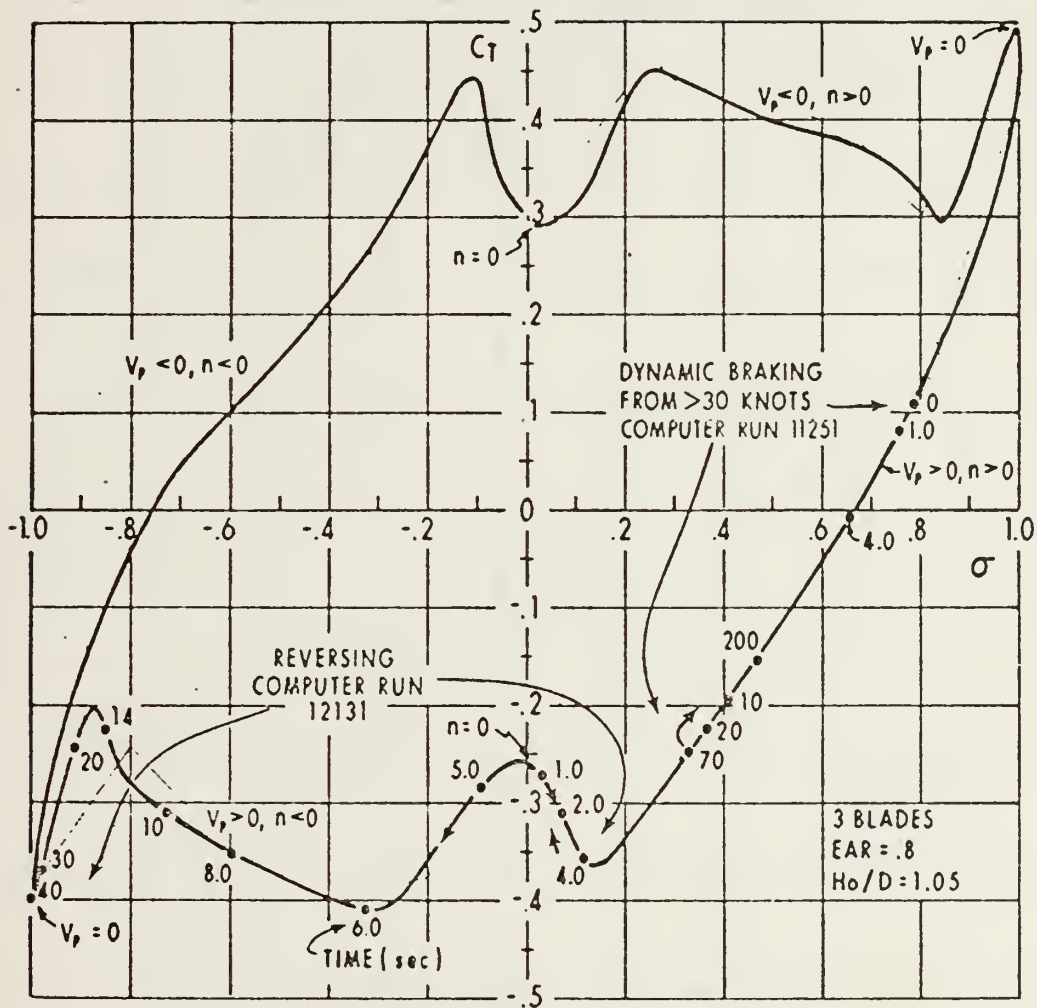


Figure B2. Thrusting Coefficient  $C_T$  versus Second Modified Advance Coefficient  $\sigma$





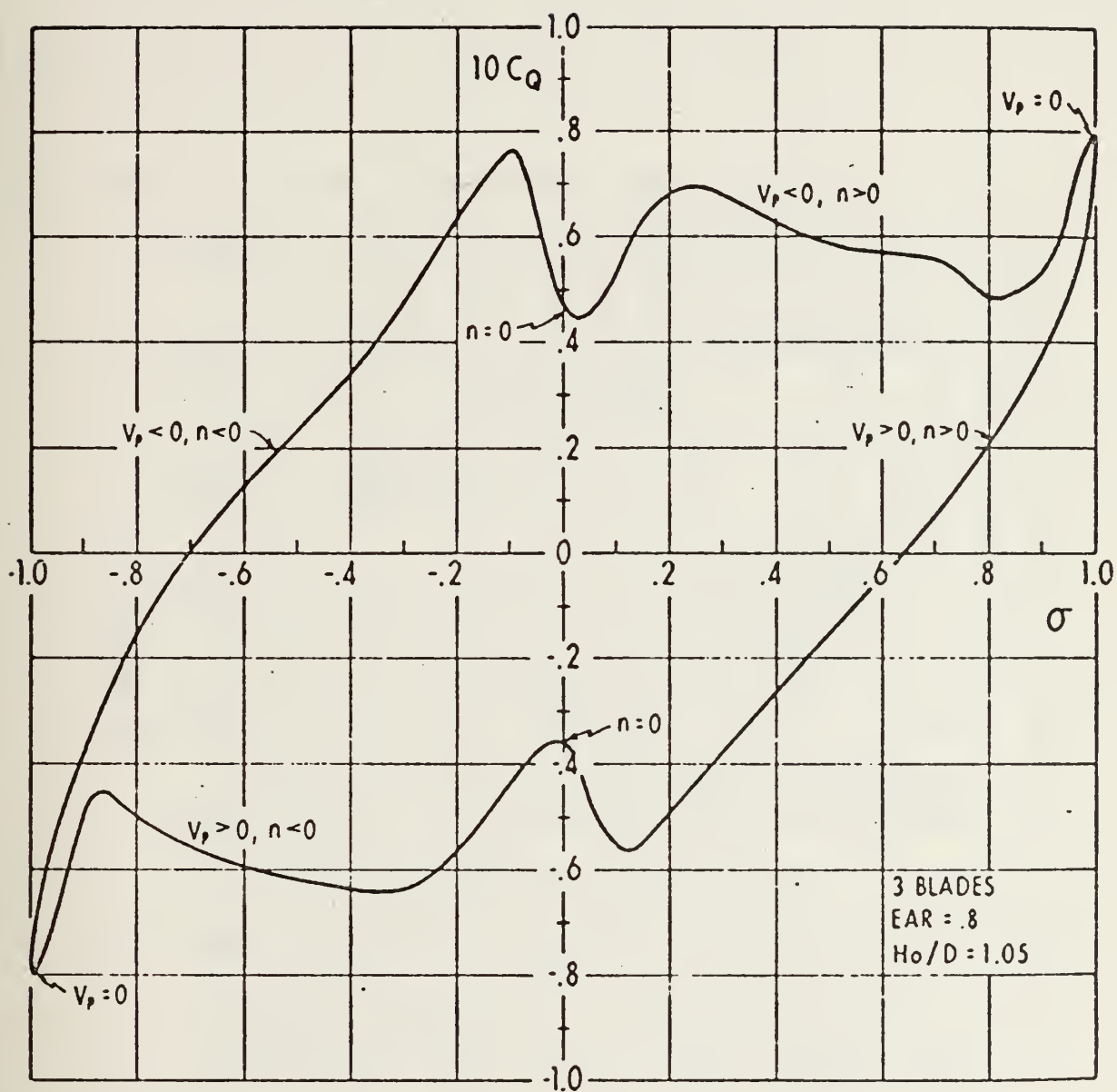


Figure B3. Torque Coefficient  $C_q$  versus Second Modified Advance Coefficient  $\sigma$ .



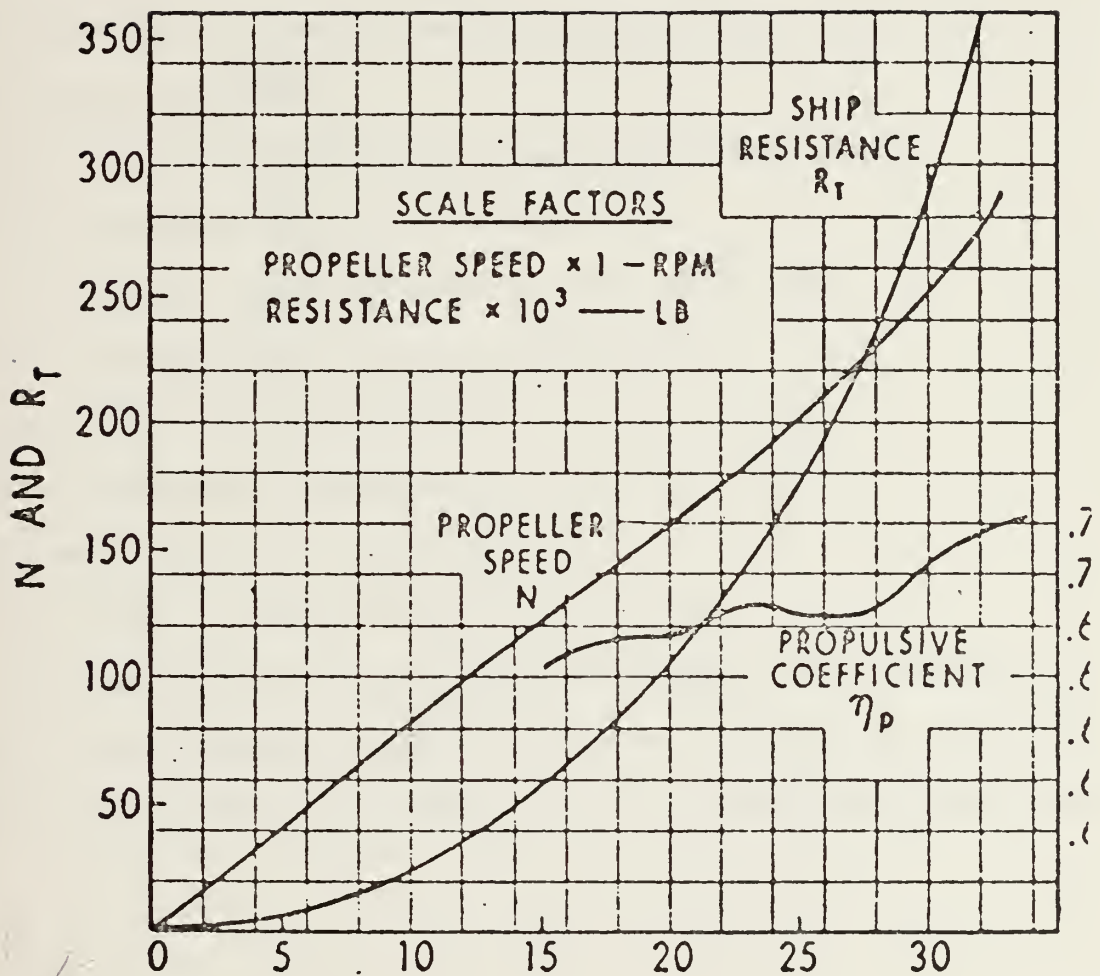


Figure B4. Ship Resistance versus Ship Speed  $V$ .



#### IV. COMBINED PROPULSION PLANT AND HULL EQUATION

##### A. POWER TURBINE WITHOUT GOVERNOR:

Nonlinear functions such as Wake fraction, Thrust deduction fraction, Thrust coefficient, torque coefficient, ship resistance and engine torque were stored in look-up tables during the computer solution.

Using the DSL/360 language these equations were solved by means of an integration and updating from table look-up every 0.1 sec until the steady state was reached.

Fig. B5. shows propeller shaft speed (rpm) versus time

Fig. B6. shows ship speed V versus time

Fig. B7. shows propeller action torque versus time

Fig. B8. shows shaft torque  $Q_e$  versus time

Fig. B9. shows  $C_q$  versus time

Fig. B10. shows  $C_t$  versus time

For a step change in fuel flow rate

$$W_f = 7000. + 3000. \text{ step } (0.0)$$

The shaft torque decreases and goes to steady state. The propeller action torque increases, overshoots at 10 sec. then goes to steady state. The propeller shaft speed increases, goes to steady state in about 80 sec.



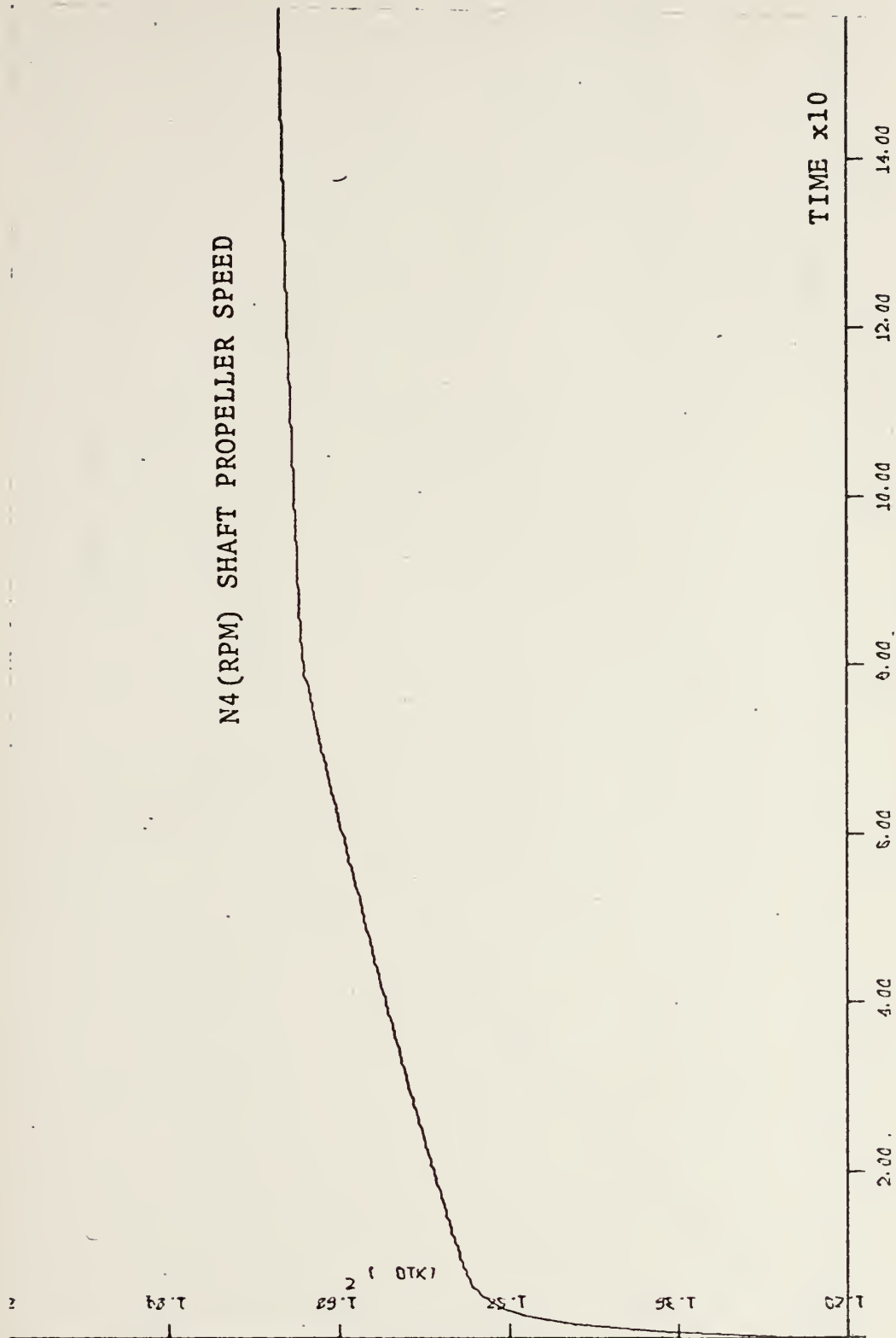


Figure B5. Shaft Propeller Speed N4 versus Time.





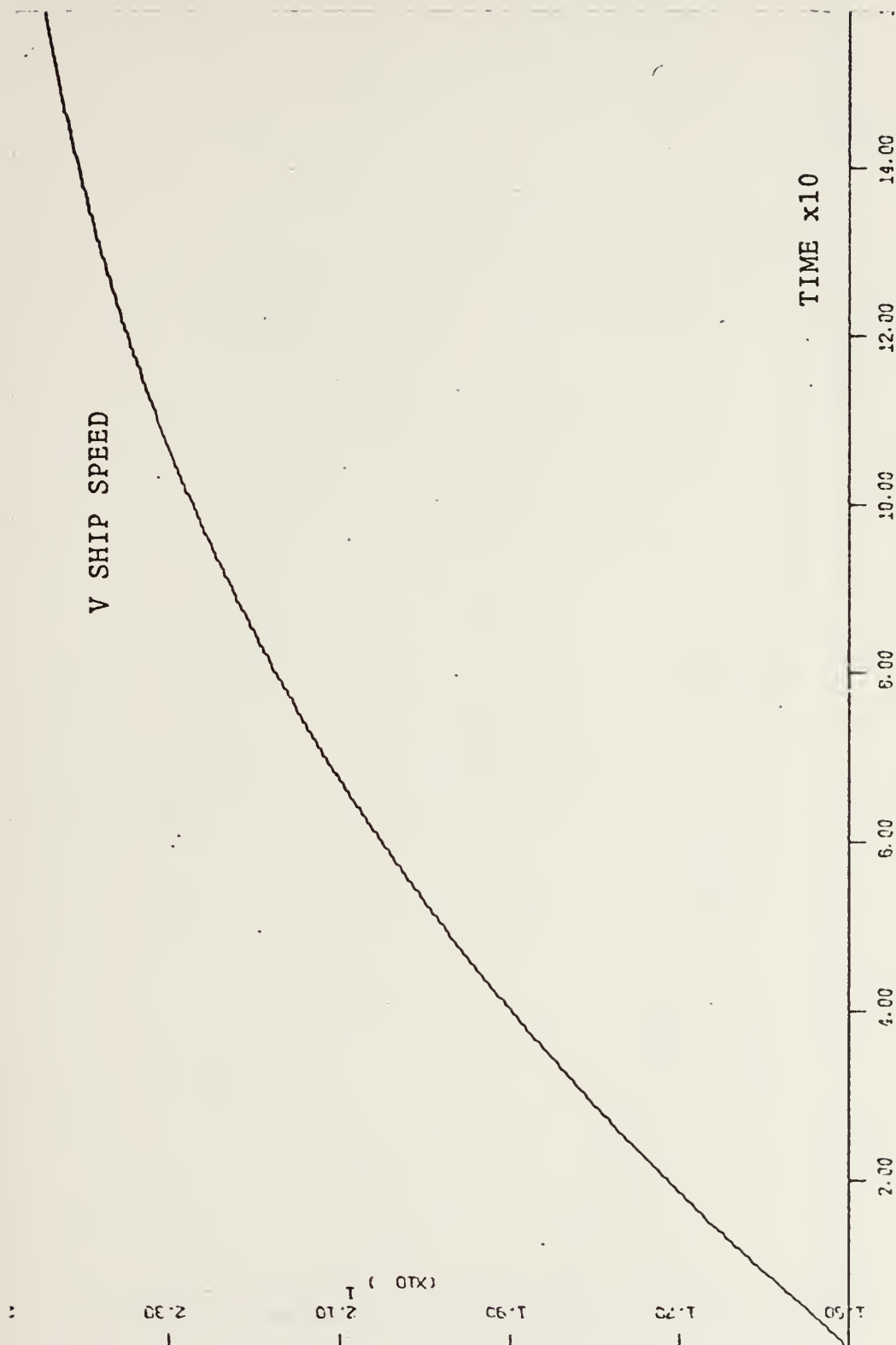


Figure B6. Ship Speed versus Time.





Figure B7. Shaft Torque versus Time.



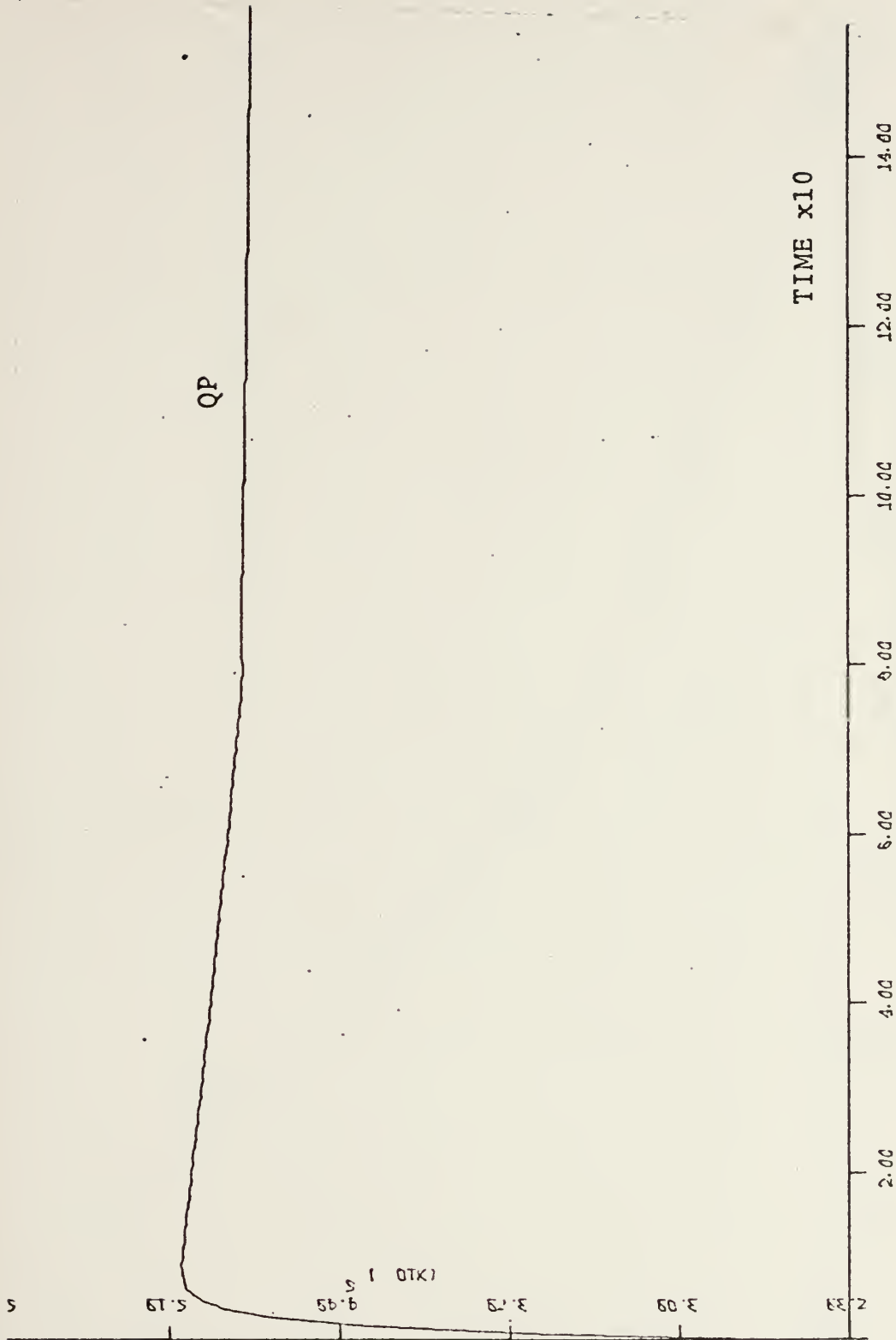


Figure B8. Propeller Reaction Torque  $Q_p$  versus Time.



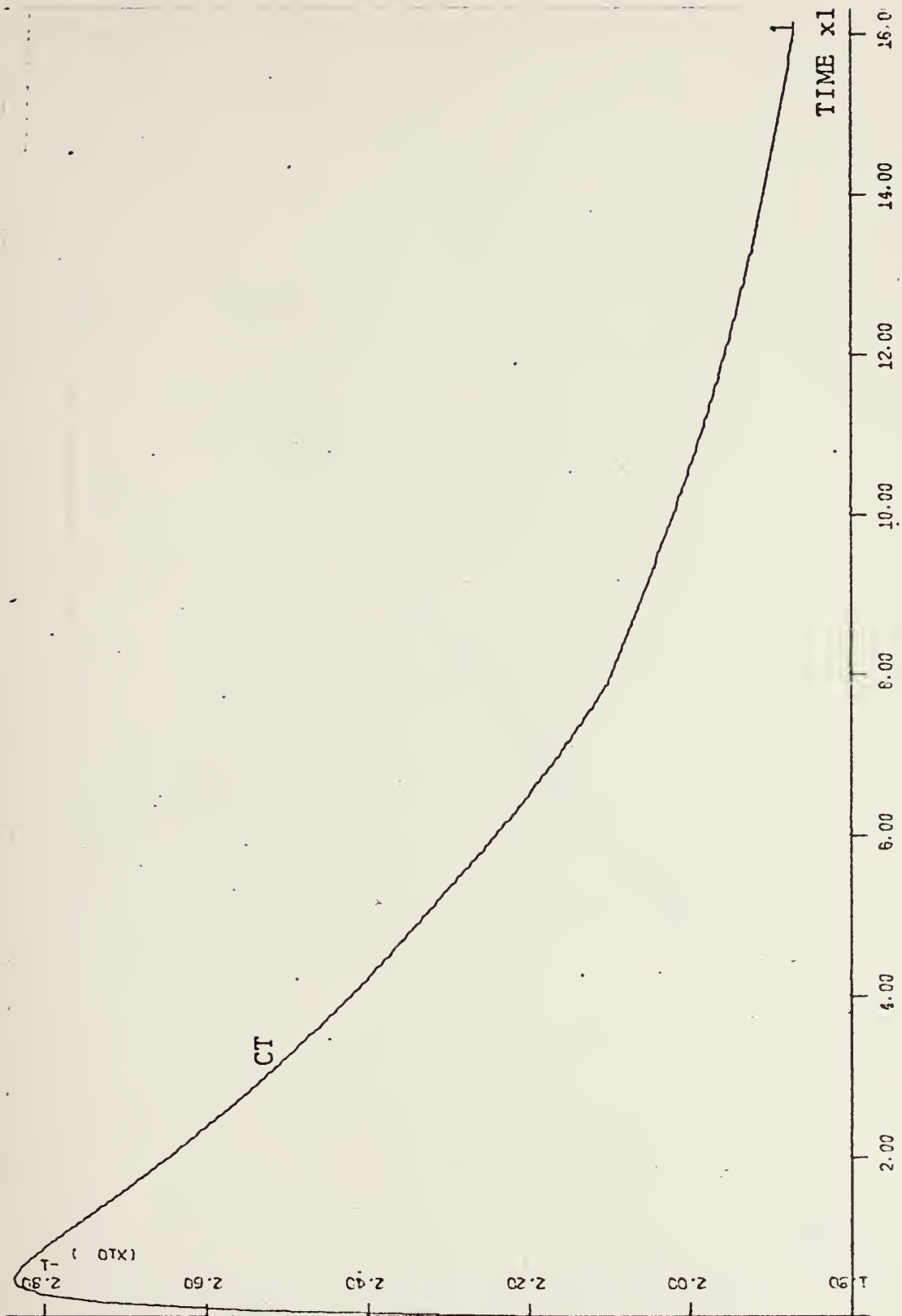


Figure B9. Thrust Coefficient Ct versus Time.





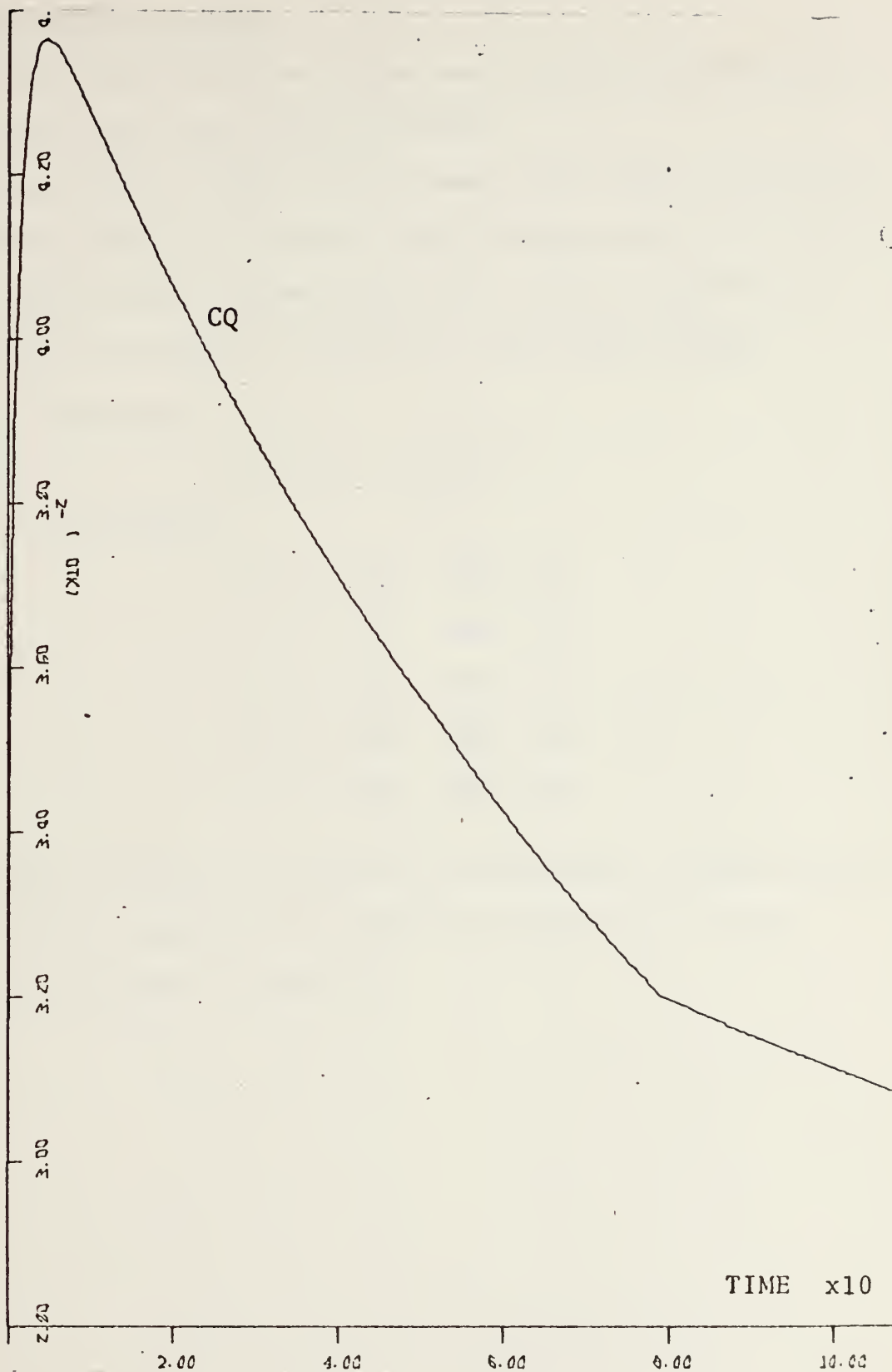


Figure B10. Torque Coefficient  $C_q$  versus Time



## B. POWER TURBINE GOVERNING:

Turbine speeds with governor are shown in Fig. A4 and compared with the responses that occur without a governor.

For small step changes in power demand at 7000 Lb/hr fuel flow rate, the governor, which has a rate of 40 Lb/hr. RPM and time lag  $1/\zeta = 1/5 = 0.2$  sec., causes the power turbine response to be reduced from 80 sec. to 20 sec. with an overshoot of 3.92% of the final change in speed.

RPM command

$$n^* = 120. + \Delta. \text{ STEP}(0.)$$

Now changing

$$\Delta = 45. \text{ RPM} \quad (\text{Fig. C1})$$

$$\Delta = 40. \text{ RPM} \quad (\text{Fig. C2})$$

$$\Delta = 30. \text{ RPM} \quad (\text{Fig. C3})$$

$$\Delta = 20. \text{ RPM} \quad (\text{Fig. C4})$$

$$\Delta = 10. \text{ RPM} \quad (\text{Fig. C5})$$

Percent Overshoot as a function of rudder angle is shown on Table 1.

The block diagram for the combined propulsion plant and hull equations were shown in Appendix B.



Increase (rpm)	N4 (MAX.)	N4 (steady state)	ERROR	Percent Overshoot
45	169.39	163.00	6.39	3.92
40	167.58	158.73	8.85	5.58
30	163.66	150.22	13.44	8.94
20	160.33	142.07	18.26	12.85
10	157.79	133.05	27.74	18.59

TABLE 1. Percent Overshoot with Governing.



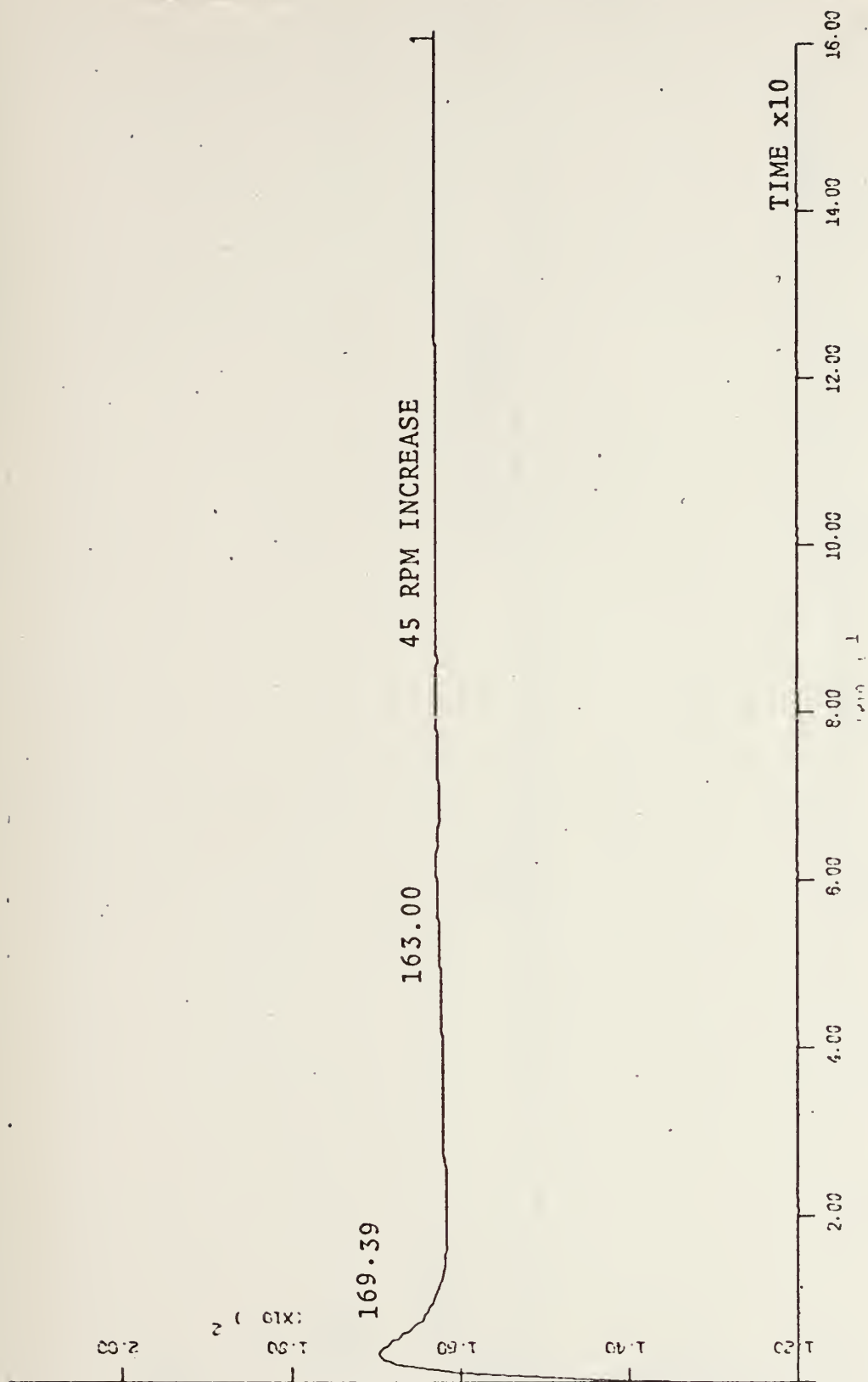
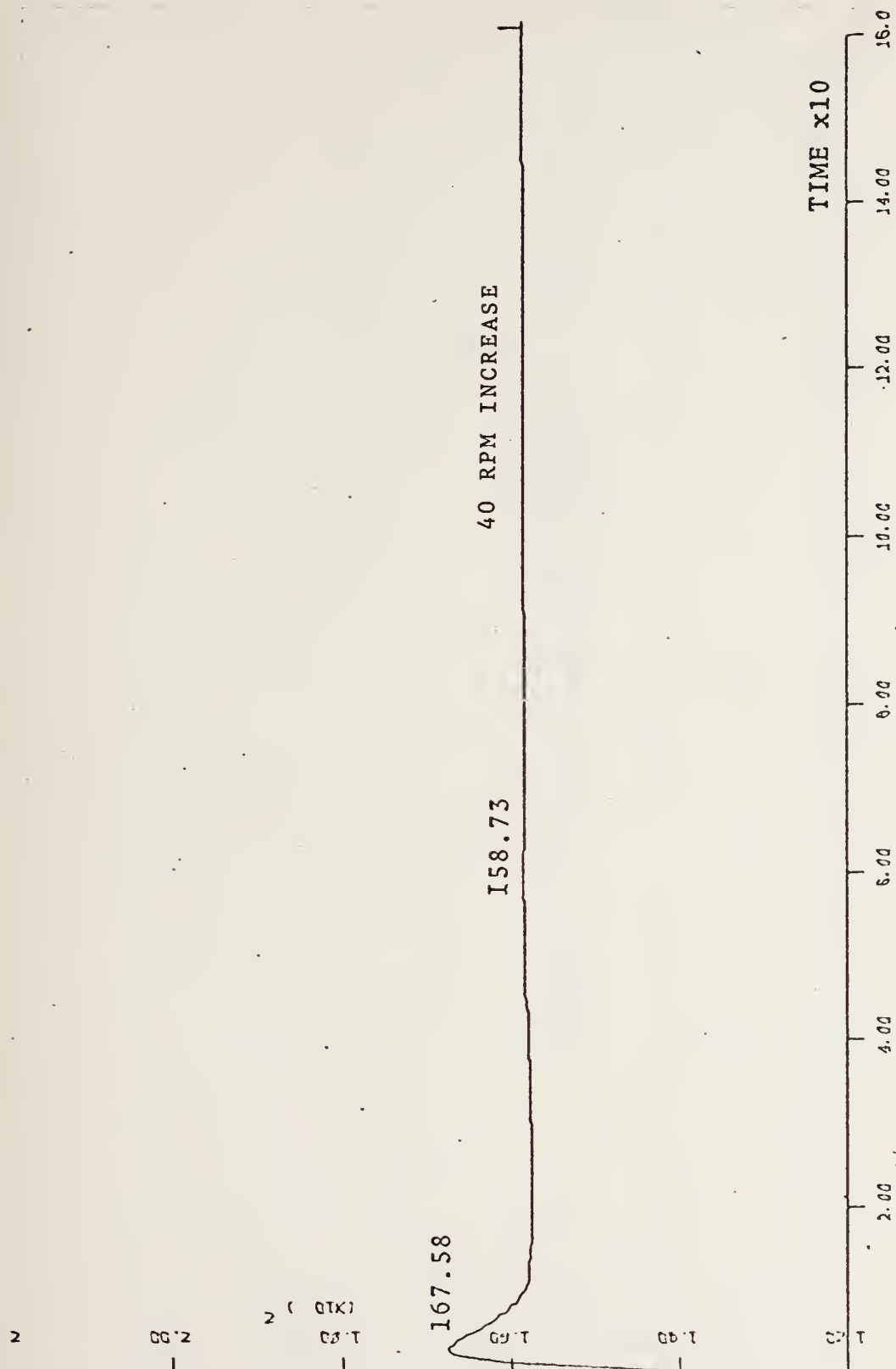


Figure C1. Shaft-Speed versus Time, with 45 RPM Increase









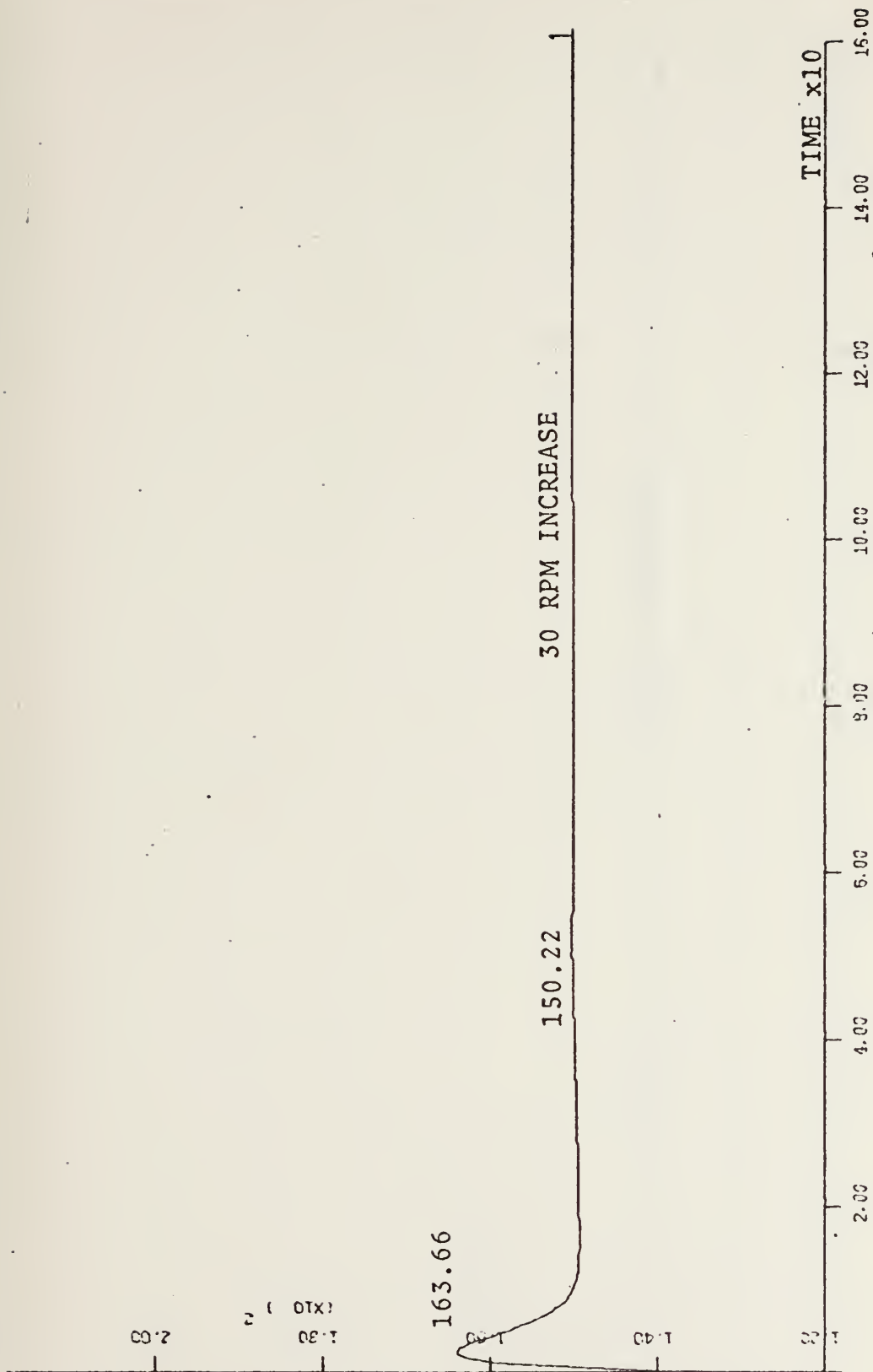


Figure C3. Shaft-Speed versus Time, with 30 RPM Increase



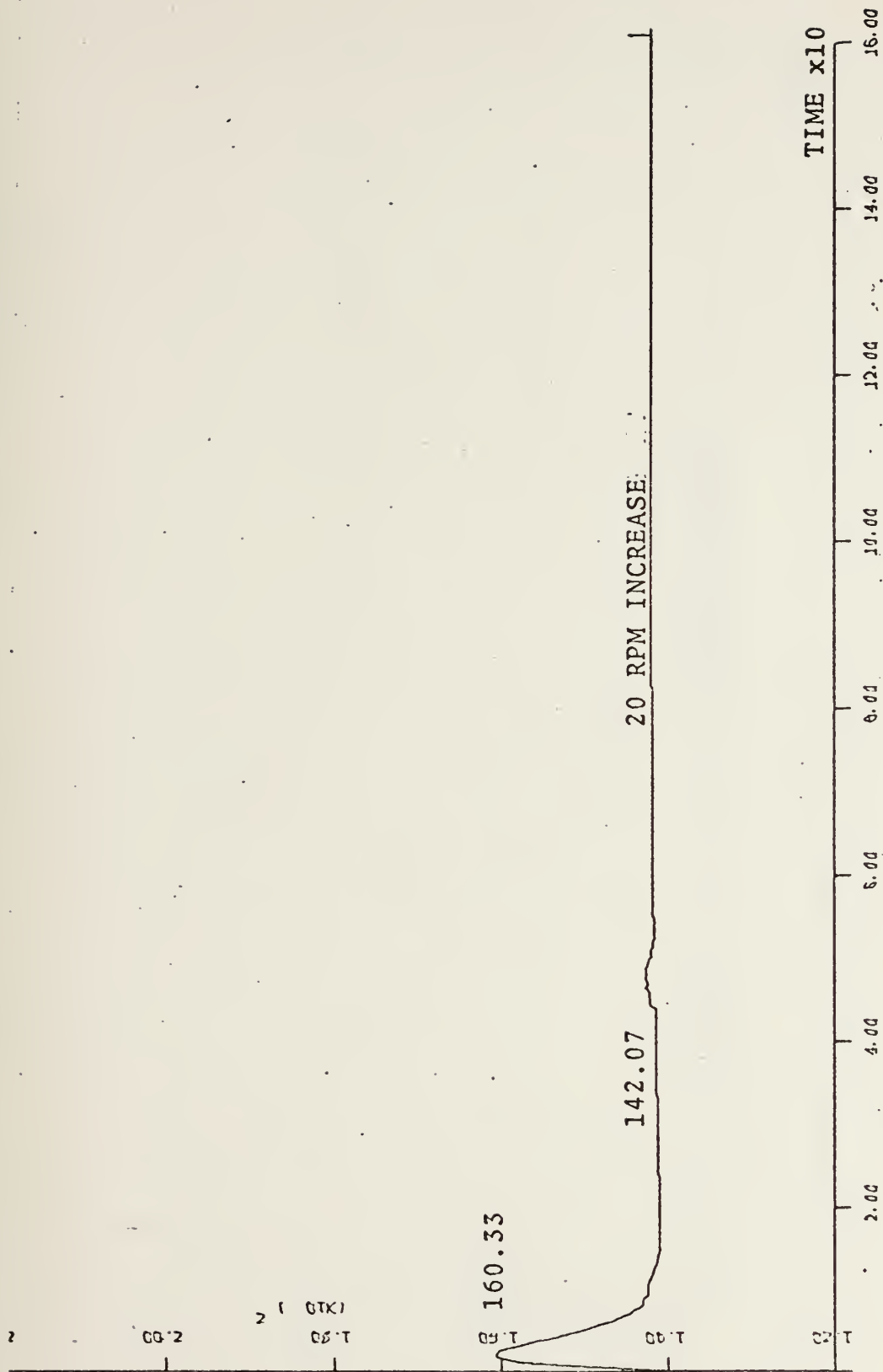


Figure C4. Shaft-Speed versus Time, with 20 RPM Increase



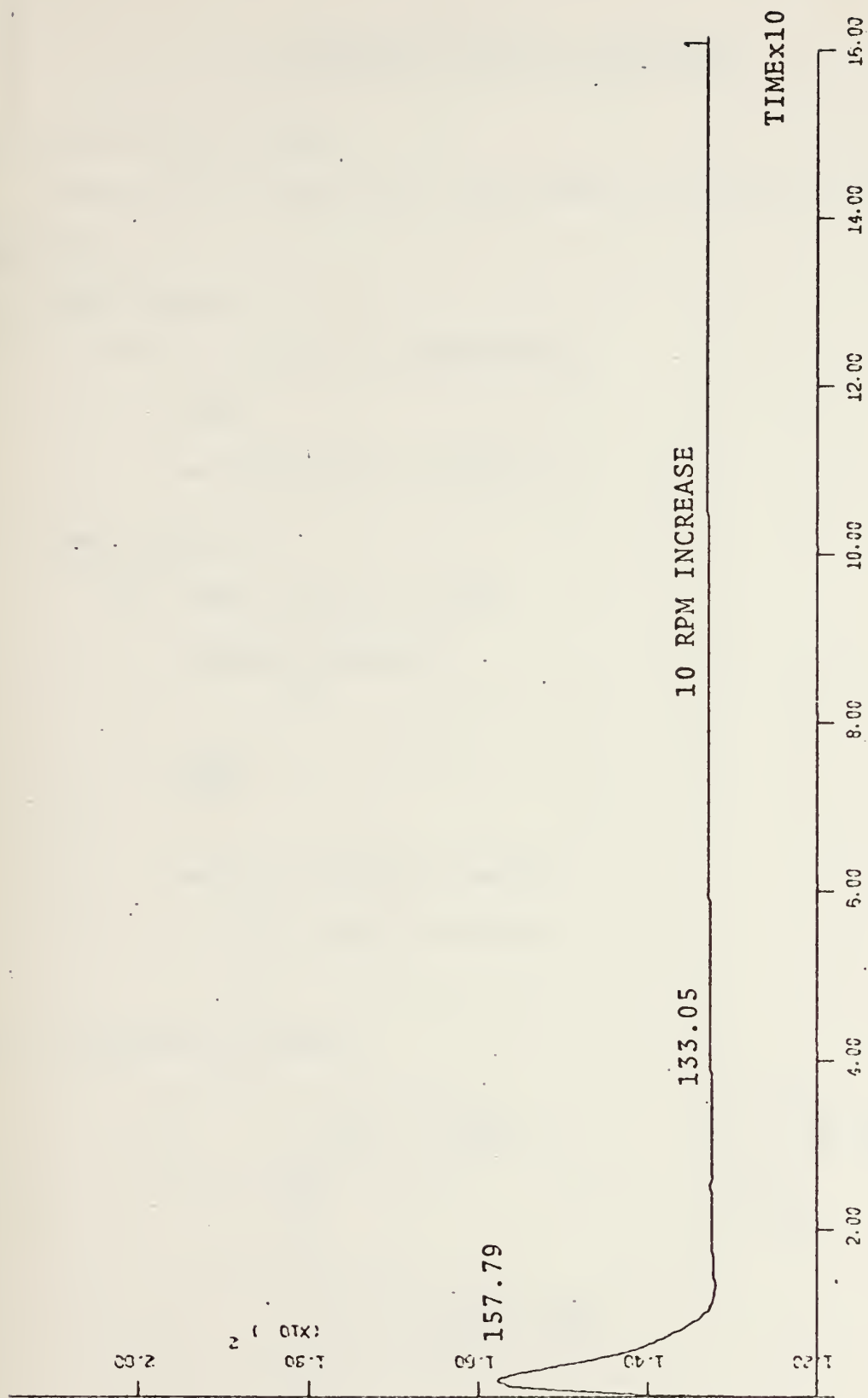


Figure C5. Shaft-Speed versus Time, with 10 RPM Increase





## V. EQUATIONS OF MOTION OF THE HULL

### A. NEWTON'S LAW OF MOTION:

Newton's Law of motion for a rigid body can be written as two equations:

#### FORCE EQUATION

$$\vec{F} = \text{Force on body} = d(\overrightarrow{\text{Momentum}})/dt$$

$$= d(m\vec{U}_G)$$

Where  $\vec{U}_G$  Velocity of body

#### MOMENT EQUATION

$$\vec{M} = \text{Moment acting on a body}$$

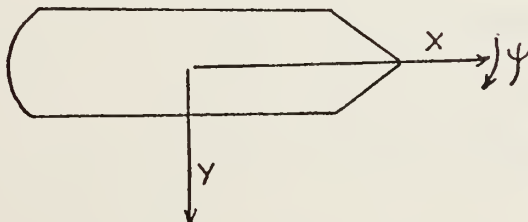
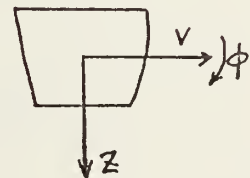
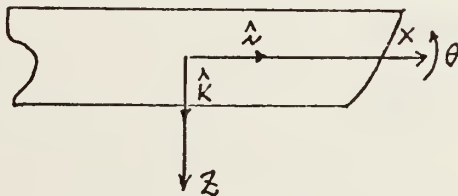
$$= \frac{d(\text{Angular momentum})}{dt}$$

$$= \frac{d(I\vec{\Omega})}{dt}$$

Where I = Moment of inertia

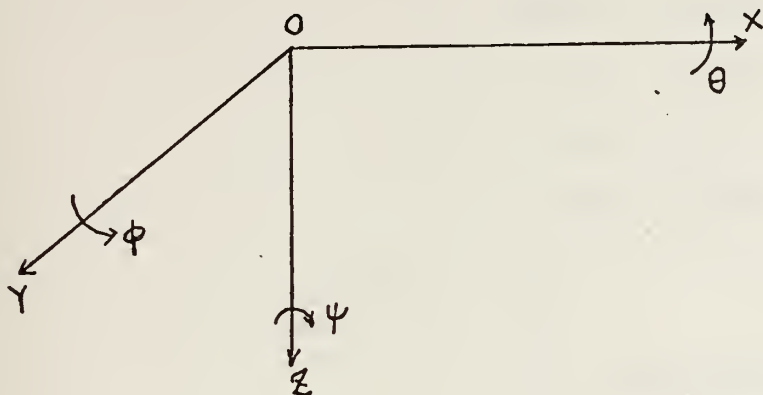
$\vec{\Omega}$  = Angular velocity

### B. SIX DEGREES OF FREEDOM:





Redrawing:



Z: Vertical axis, positive downward

Y: Tranverse axis, positive staboard

X: Longitudinal axis, positive forward

$\phi$ : Roll angle	} Positive rotations are indicated in the sketch above
$\theta$ : Pitch angle	
$\psi$ : Yaw angle	

$$\vec{F} = \frac{d(m\vec{U})}{dt} = \dot{m}\vec{U} + X(m\vec{U})$$

$$\vec{M} = \frac{d(\vec{H})}{dt} = \dot{\vec{H}} + \vec{\Omega} \times \vec{H}$$

m: Mass of ship

$$\vec{U} = u\vec{i} + v\vec{j} + w\vec{k} \quad (3)$$

}	u: Rate of surging
	v: Rate of swaying
	w: Rate of heaving



$$\vec{R} = p\mathbf{i} + q\mathbf{j} + r\mathbf{k} \quad (4)$$

R: Angular velocity

$p = \dot{\phi}$  = Rate of roll

$q = \dot{\theta}$  = Rate of yaw

$r = \dot{\psi}$  = Rate of pitch

$$\vec{F} = X\mathbf{i} + Y\mathbf{j} + Z\mathbf{k} \quad (5)$$

X: Hydrodynamic force along X axis

Y: Hydrodynamic force along Y axis

Z: Hydrodynamic force along Z axis

$\vec{M}$  = Moment vector acting on body

$$= K\mathbf{i} + M\mathbf{j} + N\mathbf{k} \quad (6)$$

K: Rolling moment about X axis

M: Pitching moment about Y axis

N: Yawing moment about Z axis

Substituting equations (3),(4),(5), and (6) into equation (2)

gives

$$X = m(\dot{u} + qw - rv - X_G(q^2 + r^2)) + Y_G(pq - \dot{r}) + Z_G(pr + \dot{q})$$

$$Y = m(\dot{v} + ru - pw - Y_G(p^2 + r^2)) + Z_G(qr - \dot{p}) + X_G(qp + \dot{r})$$

$$Z = m(\dot{w} + pv - qu - Z_G(p^2 + q^2)) + X_G(rp - \dot{q}) + Y_G(rq + \dot{p})$$

$$K = I_X \dot{p} + (I_Z - I_Y) qr - m(Y_G(\dot{w} + \dot{p}v - qu) - Z_G(\dot{v} + ru - pw))$$

$$M = I_Y \dot{q} + (I_X - I_Z) rp - m(Z_G(\dot{u} + qw - rv) - X_G(\dot{w} + pv - qu))$$

$$N = I_Z \dot{r} + (I_Y - I_X) pq - m(X_G(\dot{v} + ru - pw) - Y_G(\dot{u} + qw - rv))$$

For the surface ship in calm water, roll, pitching and heaving are taken as zero. That means  $p=q=w=0$  and most ships have  $Y_G=0$ .



Then equation (7) becomes:

$$\begin{aligned} X &= m(\dot{u} - rv - X_G r^2) \\ Y &= m(\dot{v} + ru + X_G r) \\ N &= I_Z \dot{r} - m(X_G (\dot{v} + ur)) \end{aligned} \quad (8)$$

### C. LINEARIZATION OF THE EQUATIONS OF MOTION:

#### 1. X Equation

The nonlinear TAYLOR expansion for the X equation is

$$\begin{aligned} X = X^0 &+ [X_u \Delta u + X_v v + X_r r + X_\delta \delta] \\ &+ \frac{1}{2} [X_{uu} (\Delta u)^2 + X_{vv} v^2 + X_{rr} r^2 + X_{\delta\delta} \delta^2 \\ &\quad + 2X_{uv} \Delta u v + \dots + 2X_{r\delta} r \delta] \\ &+ \frac{1}{6} [X_{uuu} (\Delta u)^3 + X_{vvv} v^3 + \dots + X_{\delta\delta\delta} \delta^3 \\ &\quad + 3X_{uuv} (\Delta u)^2 v + 3X_{uur} (\Delta u)^2 r + \dots + 3X_{r\delta\delta} r \delta^2 \\ &\quad + 6X_{uvr} (\Delta u) v r + \dots] \end{aligned} \quad (5)$$

Note:

1. The TAYLOR expansion of the equation above includes terms up to third order, since terms higher than third order are not significant.

2. Since X is an EVEN function of v, of r, of  $\delta$ .

$$X(v) = a_2 v^2 + a_4 v^4 + a_6 v^6 + \dots$$

$$X(r) = b_2 r^2 + b_4 r^4 + b_6 r^6 + \dots$$

$$X(\delta) = c_2 \delta^2 + c_4 \delta^4 + c_6 \delta^6 + \dots$$

Then the odd powers of v, r,  $\delta$  are zero i.e. all  $X_v, X_r, X_\delta$  are equal zero.





Thus equation (5) becomes:

$$X = X^0 + \frac{1}{2} X_{uu} (\Delta u)^2 + \frac{1}{2} X_{vv} v^2 + \frac{1}{2} X_{zz} z^2 + \frac{1}{2} X_{\delta\delta} \delta^2 \\ + X_{uv} (\Delta u) v + X_{uz} (\Delta u) z + X_{u\delta} (\Delta u) \delta + X_{vz} v z + X_{v\delta} v \delta + X_{z\delta} z \delta \quad (6)$$

Some of these terms are equal to zero (see Table 2)

then  $X = X_0 = 0$

## 2. Y Equation

The nonlinear TAYLOR expansion for the Y equation is:

$$Y = Y^0 + \left[ Y_u \Delta u + Y_v v + Y_z z + Y_\delta \delta \right] \\ + \frac{1}{2} \left[ Y_{uu} (\Delta u)^2 + Y_{vv} v^2 + Y_{zz} z^2 + Y_{\delta\delta} \delta^2 + 2 Y_{uz} \Delta u \cdot z + \dots \right. \\ \left. + 2 Y_{z\delta} z \delta \right] + \\ + \frac{1}{6} \left[ Y_{uuu} (\Delta u)^3 + Y_{vvv} v^3 + Y_{zzz} z^3 + Y_{\delta\delta\delta} \delta^3 \right. \\ + 3 Y_{uuv} (\Delta u)^2 v + 3 Y_{uuz} (\Delta u)^2 z + \dots + 3 Y_{z\delta\delta} z \delta^2 \\ \left. + 6 Y_{uvz} \Delta u v z + \dots \right] \quad (7)$$

NOTE:

1. The terms higher than third order are not significant.
2. Y is an odd function of v, r,  $\delta$

$$Y(v) = d_1 v + d_3 v^3 + d_5 v^5 + \dots$$

$$Y(r) = e_1 r + e_3 r^3 + e_5 r^5 + \dots$$

$$Y(\delta) = f_1 \delta + f_3 \delta^3 + f_5 \delta^5 + \dots$$



Then equation (7) becomes:

$$\begin{aligned}
 Y = & Y^0 + Y_u \Delta u + Y_v v + Y_r r + Y_\delta \delta + \\
 & + \frac{1}{6} Y_{uuu} (\Delta u)^3 + \frac{1}{6} Y_{vvv} v^3 + \frac{1}{6} Y_{rrr} r^3 + \frac{1}{6} Y_{\delta\delta\delta} \delta^3 \\
 & + \frac{1}{2} Y_{uuv} (\Delta u)^2 v + \frac{1}{2} Y_{uur} (\Delta u)^2 r + \frac{1}{2} Y_{u\delta\delta} (\Delta u)^2 \delta + \frac{1}{2} Y_{rvv} r v^2 + \frac{1}{2} Y_{\delta r r} \delta r^2 \\
 & + \frac{1}{2} Y_{v\delta\delta} v \delta^2 + \frac{1}{2} Y_{r\delta\delta} r \delta^2 + Y_{v r \delta} v r \delta
 \end{aligned}$$

Where the indicated terms are equal to zero (see Table 3)

$$Y = Y_0 + Y_v v + Y_r r + Y_\delta \delta + \frac{1}{6} Y_{vvv} v^3 + \frac{1}{6} Y_{\delta\delta\delta} \delta^3 + \frac{1}{2} Y_{rvv} r v^2 + \frac{1}{6} Y_{rrr} r^3 \quad (8)$$

### 3. N Equation

The nonlinear TAYLOR expansion for the N equation is:

$$\begin{aligned}
 N = & N^0 + N_u \Delta u + N_v v + N_r r + N_\delta \delta + \\
 & + \frac{1}{2} \left[ N_{uu} (\Delta u)^2 + N_{vv} v^2 + N_{rr} r^2 + N_{\delta\delta} \delta^2 + \right. \\
 & \quad \left. + 2 N_{ur} (\Delta u) r + \dots + 2 N_{r\delta} r \delta \right] \\
 & + \frac{1}{6} \left[ N_{uuu} \Delta u^3 + N_{vvv} v^3 + N_{rrr} r^3 + N_{\delta\delta\delta} \delta^3 \right. \\
 & \quad + 3 N_{uuv} (\Delta u)^2 v + 3 N_{uur} (\Delta u)^2 r + \dots + 3 N_{r\delta\delta} r \delta^2 \\
 & \quad \left. + 6 N_{uvr} \Delta u \cdot v r + \dots \right] \quad (9)
 \end{aligned}$$

NOTE:

1. The terms higher than third order are not significant.
2. N is an odd function of v, r,  $\delta$

$$N(v) = g_1 v + g_3 v^3 + g_5 v^5 + \dots$$

$$N(r) = h_1 r + h_3 r^3 + h_5 r^5 + \dots$$

$$N(\delta) = K_1 \delta + K_3 \delta^3 + K_5 \delta^5 + \dots$$



Then equation (9) becomes:

$$N = N_u \Delta u + N_v v + N_z z + N_\delta \delta + \\ \frac{1}{6} N_{uuu} (\Delta u)^3 + \frac{1}{6} N_{vvv} v^3 + \frac{1}{6} N_{zzz} z^3 + \frac{1}{6} N_{\delta\delta\delta} \delta^3 \\ + \frac{1}{2} N_{uvv} (\Delta u)^2 v + \frac{1}{2} N_{uuu} \Delta u^2 z + \dots + N_{vz\delta} v z \delta$$

The indicated terms are equal to zero (see Table 4)

Then

$$N = N_v v + N_z z + N_\delta \delta + \frac{1}{6} N_{vvv} v^3 + \frac{1}{6} N_{zzz} z^3 + \frac{1}{2} N_{vzz} v z^2 + \frac{1}{2} N_{zvw} z v^2 \\ (10)$$

Combining the third order TAYLOR expansion for X,Y, and N with the dynamic response terms of the X,y, and N equation.

X equation       $X = 0$

Y equation

$$(m - Y\dot{v})\dot{v} + (mX_G - Y\dot{r})\dot{r} = Y(u, v, r, \delta)$$

N equation

(1)

$$(mX_G - N\dot{v})\dot{v} + (I_Z - N\dot{r})\dot{r} = N(u, v, r, \delta)$$



Using CRAMER'S rule to solve these equations

$$\dot{V} = \frac{\begin{vmatrix} Y(u,v,r,\delta) & mX_G - Y_r \\ N(u,v,r,\delta) & I_Z - N_r \end{vmatrix}}{\begin{vmatrix} m - Y_v & mX_G - Y_r \\ mX_G - N_v & I_Z - N_r \end{vmatrix}}$$

$$\dot{r} = \frac{\begin{vmatrix} m - Y_v & Y(u,v,r, ) \\ mX_G - N_v & N(u,v,r, ) \end{vmatrix}}{\begin{vmatrix} m - Y_v & mX_G - Y_r \\ mX_G - N_v & I_Z - N_r \end{vmatrix}}$$





TABLE 2. ASSESSMENT OF THE COEFFICIENTS IN  
THE X EQUATION

Variable	Coefficient	Series 60/Model 5.1.1	
$\dot{u}$	$m-X\dot{u}$	0.1750	
$\Delta u$	$Xu$	—	
$\Delta u^2$	$1/2 X_{uu}$	—	
$\Delta u^3$	$1/6 X_{uuu}$	—	
$v^2$	$1/2 X_{vv}$	—	
$r^2$	$1/2 X_{rr} + mX_G$	—	
$\delta^2$	$1/2 X_{\delta\delta}$	—	
$v^2\Delta u$	$1/2 X_{vvu}$	—	
$r^2\Delta u$	$1/2 X_{rru}$	—	
$\delta^2\Delta u$	$1/2 X_{\delta\delta u}$	—	
$v\dot{r}$	$X_{vr} + m$	—	
$v\delta$	$X_{v\delta}$	—	
$r\delta$	$X_{r\delta}$	—	
$v\dot{r}\Delta u$	$X_{vru}$	—	
$v\delta\Delta u$	$X_{v\delta u}$	—	
$r\delta\Delta u$	$X_{r\delta u}$	—	
—	$X_o$	—	

Note: No entry in these columns means the coefficient was ignored.

\* From PRINCIPLES OF NAVAL ARCHITECTURE, SNAME, 1968, page 548.



TABLE 3. ASSEMBLY OF THE COEFFICIENTS IN  
THE Y EQUATION

Variable	coefficient	Series 60/5.1.	
$\dot{v}$	$m-Y\dot{v}$	0.309	Y9
$\dot{r}$	$mX_G-Y\dot{r}$	—	Y8
$v$	$Yv$	-0.260	Y1
$v^3$	$1/6 Yvvv$	-2.150	Y2
$vr^2$	$1/2 Yvrr$	-1.180	Y3
$v\delta^2$	$1/2 Yv\delta\delta$	—	
$v\Delta u$	$Yvu$	—	
$v\Delta u^2$	$1/2 Yvu u$	—	
$r$	$(Yr-m)$	-0.0781	YY4
$r^3$	$1/6 Yrrr$	-0.0461	Y5
$rv^2$	$1/2 Yrvv$	-0.0994	Y6
$r\delta^2$	$1/2 Yr\delta\delta$	—	
$r\Delta u$	$Yru$	—	
$r\Delta u^2$	$Yru u$	—	
$\delta$	$1/6 Y\delta$	<u>0.050</u>	Y7
$\delta v^2$	$1/2 Y\delta rr$	—	
$\delta\Delta u$	$Y\delta u$	—	
$\delta\Delta u^2$	$1/2 Y\delta u u$	—	
$vr\delta$	$Yvr\delta$	—	
—	$Y_0$	0.00016	Y <sub>0</sub>
$\Delta u$	$YY_u^0$	—	
$(\Delta u)^2$	$Y_{uu}^0$	—	

\*No entry in these columns means the coefficient was ignored.

\*From PRINCIPLES OF NAVAL ARCHITECTURE, SNAME, 1968, page 548.



TABLE 4. ASSESSMENT OF THE COEFFICIENT IN  
THE N EQUATION.

Variable	coefficient	Series 60/Model 5.1.1	
$\dot{v}$	$mX_G - N\dot{v}$	_____	N9
$\dot{r}$	$I_Z - Nr\dot{r}$	0.0202	N8
$v$	$Nv$	-0.0750	N1
$v^3$	$1/6 Nvvv$	-0.3850	N2
$vr^2$	$1/2 Nvrr$	-0.3060	N3
$v\delta^2$	$1/2 Nv\delta\delta$	_____	
$v\Delta u$	$Nvu$	_____	
$v\Delta u^2$	$1/2 Nvu u$	_____	
$r$	$(Nr - mX_G)$	-0.0569	NN4
$r^3$	$1/6 Nrrrr$	-0.1010	N5
$rv^2$	$1/2 Nr v v$	-1.4200	N6
$r\delta^2$	$1/2 Nr\delta\delta$	_____	
$r\Delta u$	$Nru$	_____	
$r\Delta u^2$	$1/2 Nru u$	_____	
$\delta$	$N\delta$	-0.0240	N7
$\delta^3$	$1/6 N\delta\delta\delta$	_____	
$\delta v^2$	$1/2 N\delta v v$	_____	
$\delta r^2$	$1/2 N\delta r r$	_____	
$\delta\Delta u$	$N\delta u$	_____	
$\delta\Delta u^2$	$1/2 N\delta u u$	_____	
$vr\delta$	$Nvr\delta$	_____	
—	No	-0.0003	No
$\Delta u$	$N_u^o$	_____	
$(\Delta u)^2$	$N_{uu}^o$	_____	

\*No entry in these columns means the coefficient was ignored.

\*From PRINCIPLES OF NAVAL ARCHITECTURE, SNAME, 1968, page 549.



#### D. RELATIONS BETWEEN FIXED AXES AND MOVING AXES:

So far the reference axes  $X, Y$ , and  $Z$  are fixed to the moving ship. In order to determine the path of the ship, it is convenient to take the fixed axes to the earth  $X_0, Y_0$ .

In figure D1., the ship is at the position A.

The YAW ANGLE  $\psi$  is the angle between the  $U$  axis and the tangent line with the path at point A.

Then the following relationships between the fixed and the moving axes can be written:

$$X_0(t) = U(t) \cos \psi(t) - V(t) \sin \psi(t)$$

$$Y_0(t) = U(t) \sin \psi(t) + V(t) \cos \psi(t)$$

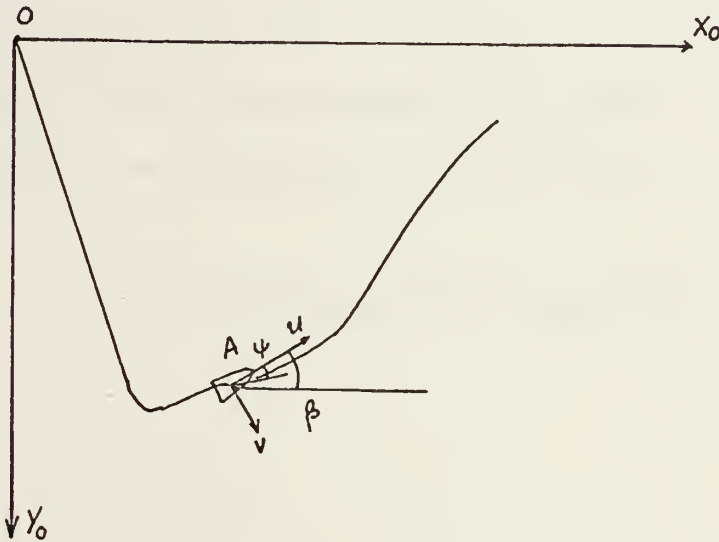


Figure D1. Relations Between Fixed Axes in the Earth and the Ship Axes.





E. RUDDER DEFLECTION:

Let  $\delta$  = Rudder angle

DRATE = Rudder deflection rate.

TDMAX = Time at maximum rudder deflection

Then the rudder deflection in turning maneuver can be shown in

Figure D2:

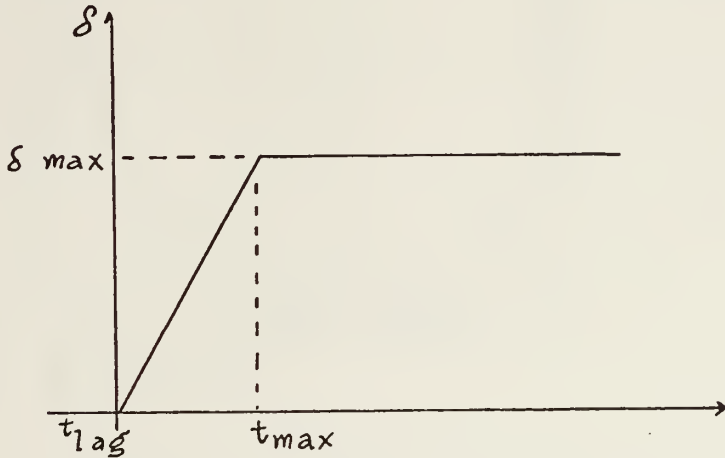


Figure D2. Rudder Deflection versus Time.

Let Rudder deflection =  $\delta$

DRATE = Rudder deflection rate =  $D_{max}/TDMAX$

Then the function of the rudder deflection can be written:

$$\delta = \text{DRATE} \left[ \text{RAMP}(t_{lag}) - \text{RAMP}(t - t_{max}) \right] \frac{1}{57.3}$$



## F. THE TURNING PATH OF A SHIP:

There are 4 phases of a turn:

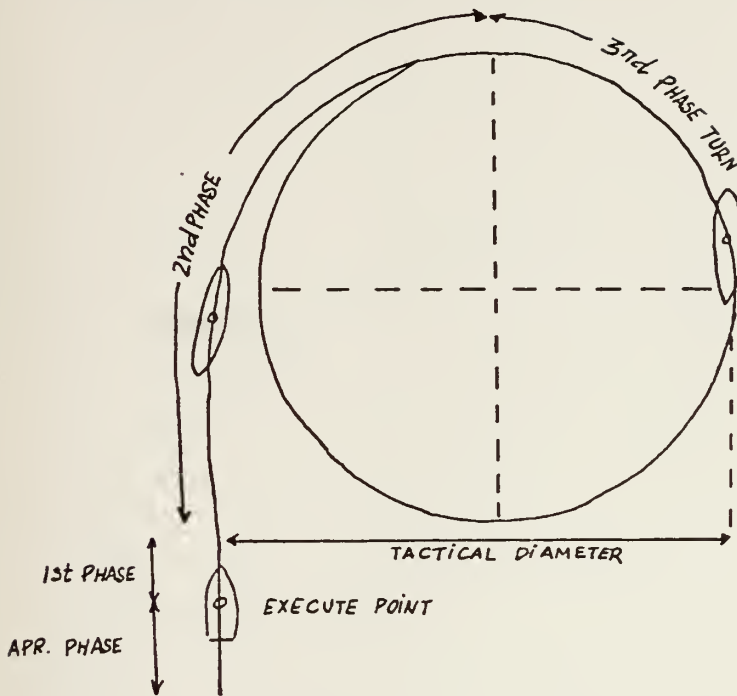


Figure D3. The Turning Path of a Ship.

Approach phase	$\dot{v} = 0, \dot{r} = 0, v = 0, r = 0$
First phase of turn	$\dot{v} \neq 0, \dot{r} \neq 0, v = 0, r = 0$
Second phase of turn	$\dot{v} \neq 0, \dot{r} \neq 0, v \neq 0, r \neq 0$
Third phase of turn	$\dot{v} = 0, \dot{r} = 0, v \neq 0, r \neq 0$

Using dimensional equation (1) and introducing the rudder deflection  $30^\circ$ . The characteristics of transient phases of a turn are shown in Figures D6, D7, D8, and D9.

A-B Second phase	$v \neq 0, \dot{v} \neq 0, r \neq 0, \dot{r} \neq 0$
B-C Third phase	$v \neq 0, \dot{v} = 0, r \neq 0, \dot{r} \neq 0$

Now using different rudder deflections  $=35^\circ, 30^\circ$  the turning path of a ship is shown in Figures D4 and D5. When using the bigger deflection the steady turning radius is smaller.



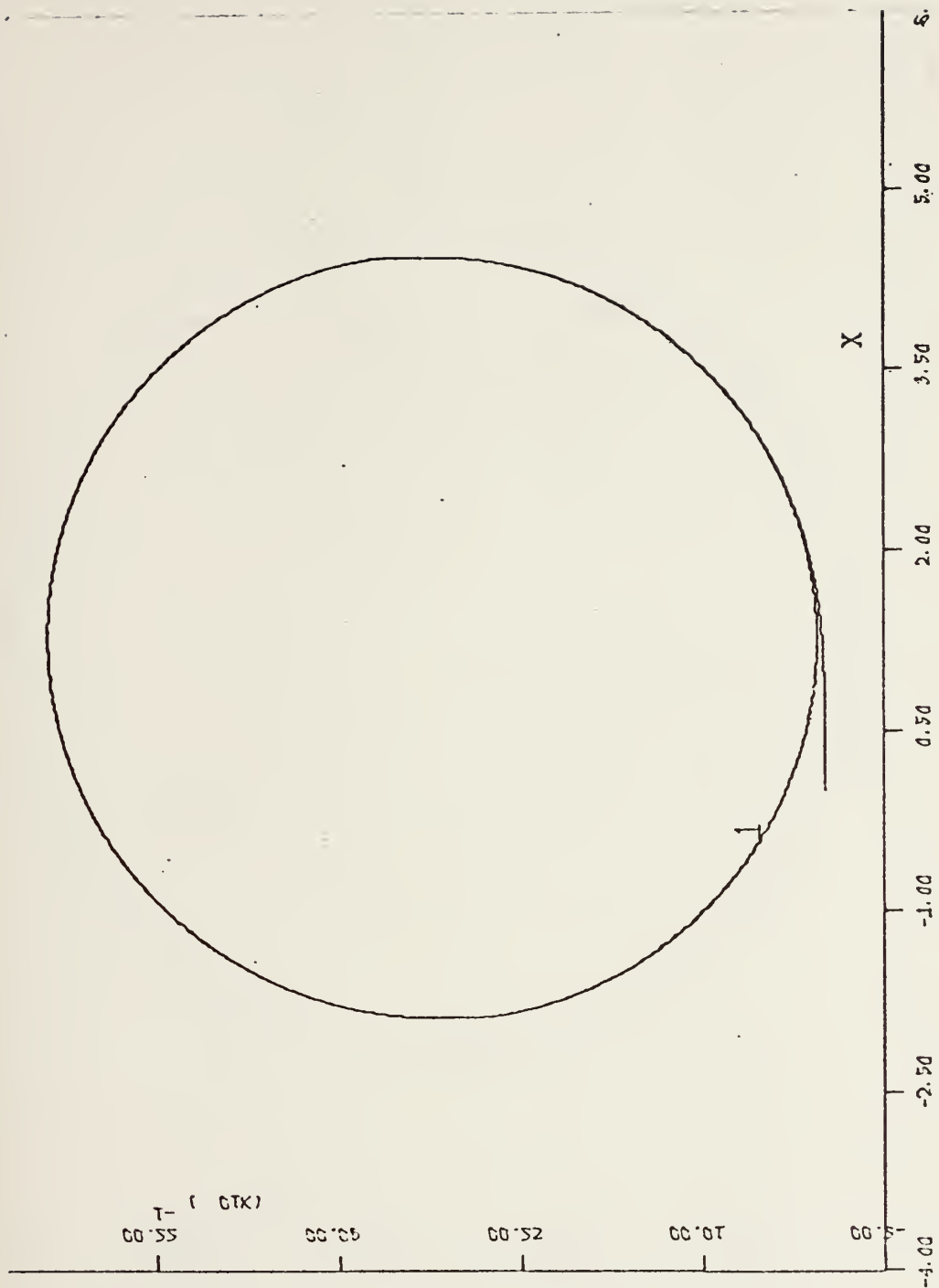


Figure D4. The Turning Path of a Ship with 35 Degree Rudder Angle.



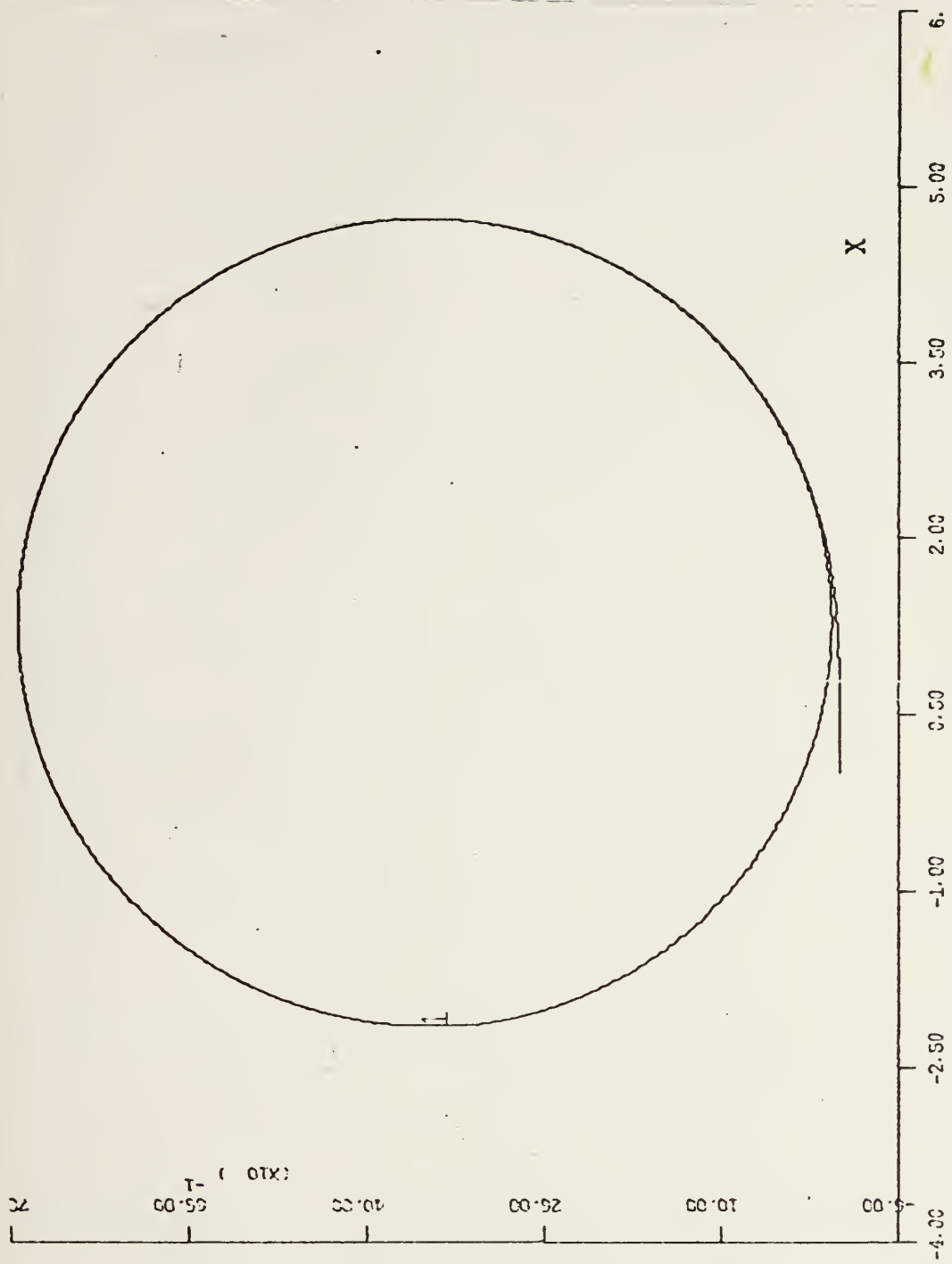


Figure D5. The Turning Path of a Ship with 30 Degree Rudder Angle.





TUAN

RDOT

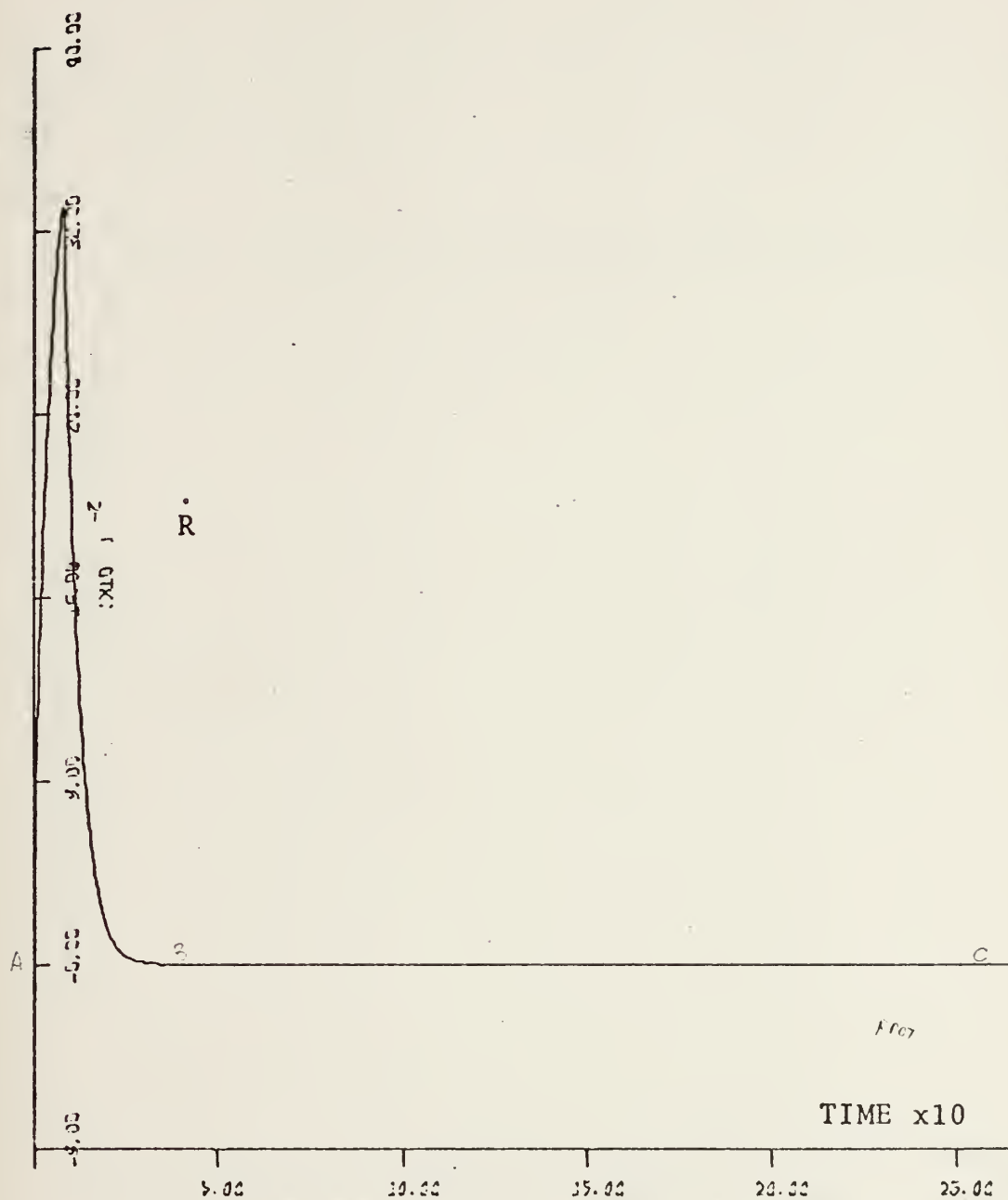


Figure D6. Angular Acceleration about Z Axis.



TUAN

R

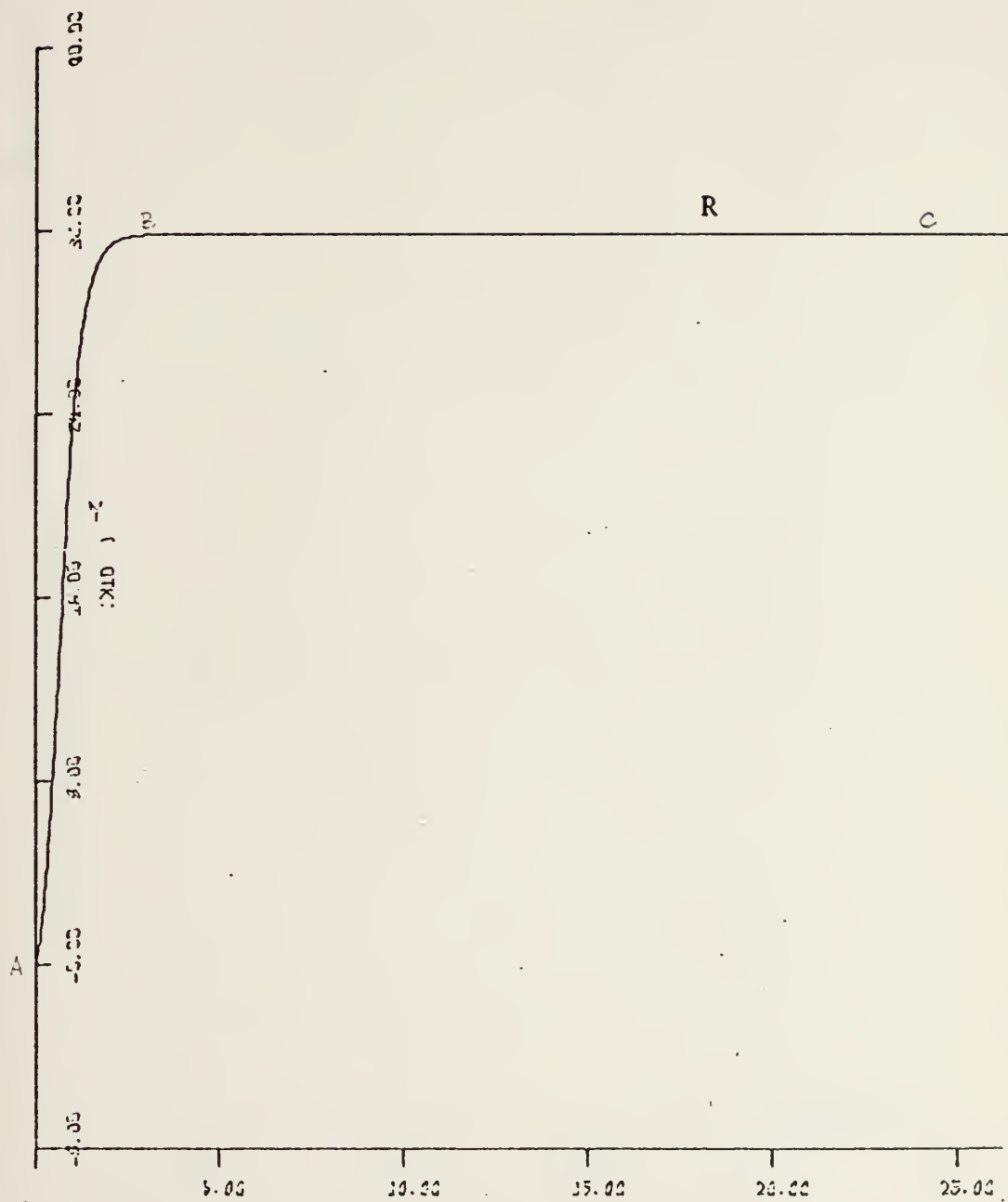


Figure D7. Angular Velocity about Z Axis.



TUAN

VDOT

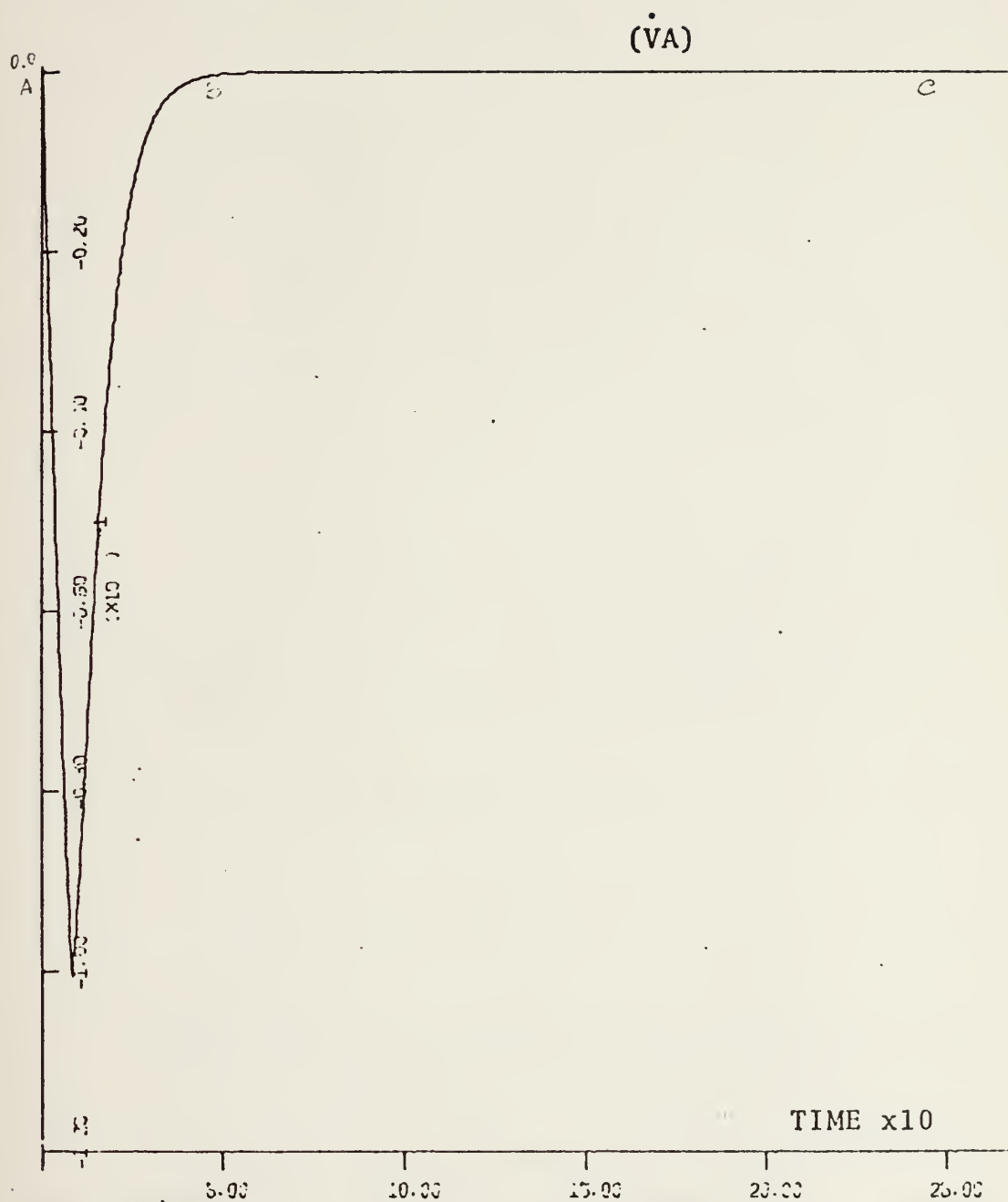


Figure D8. Angular Acceleration about Y Axis.



TUAN

VA

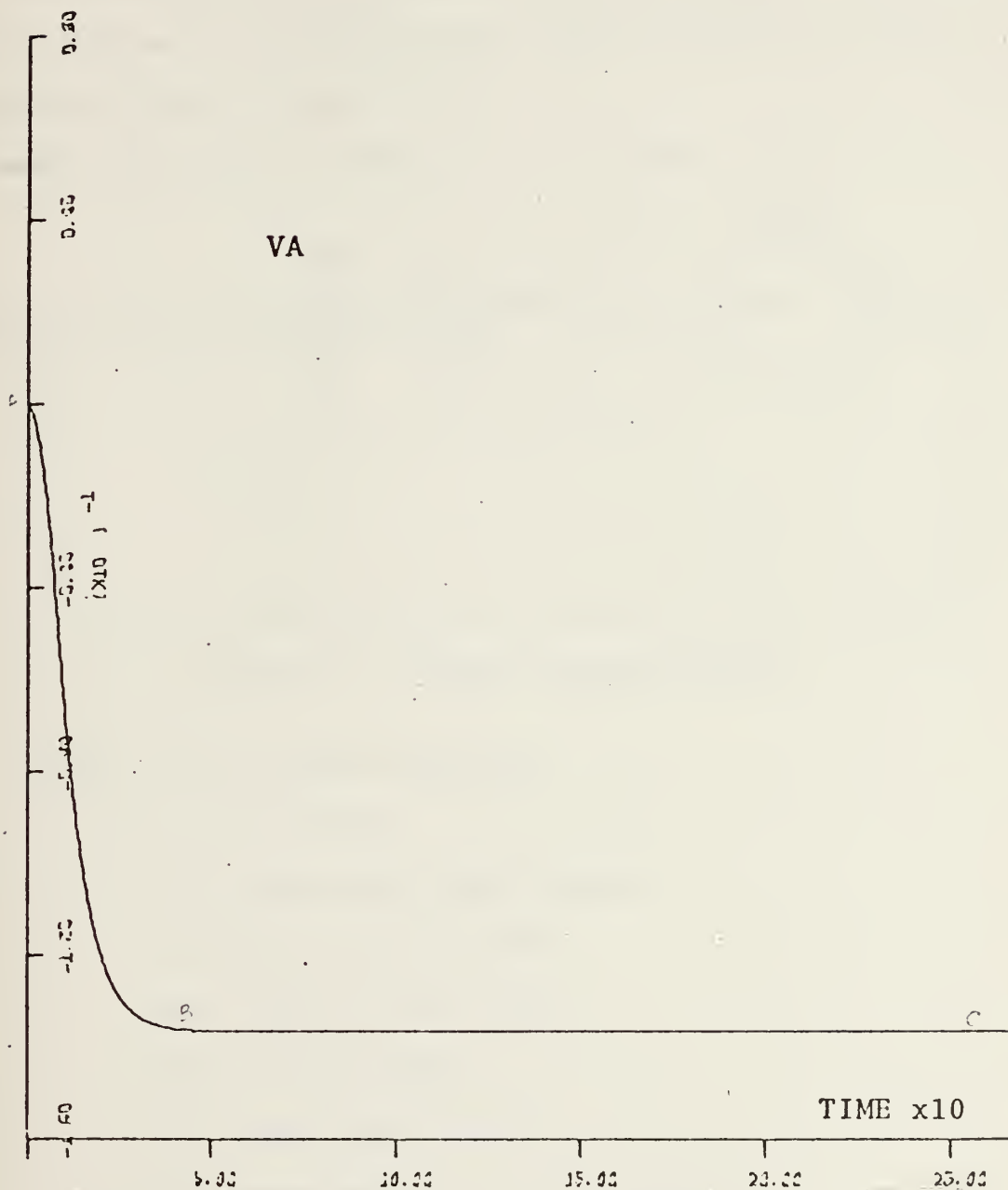


Figure D9. Angular Velocity about Y Axis.





## VI. SPEED CONTROL IN TURNS

### A. DYNAMIC BEHAVIOR OF THE PROPULSION PLANT:

The propulsion plant and turning path were observed. Now combine both of them and add feed-back of ship speed with Gain  $K_1$  in order to keep the ship speed constant in turns.

The block diagram for this part can be drawn:

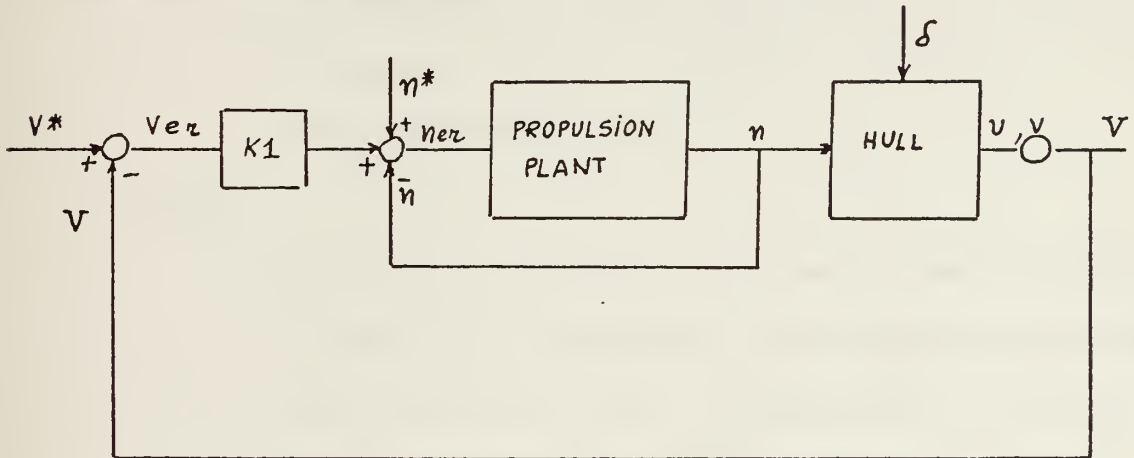


Figure E1. Feedback Control Velocity.

Where:  $V^*$ : Reference ship speed.

$V$  : Ship speed.

$n^*$ : Shaft angular speed command.

$U$  : Velocity about X axis.

$V_A$ : Velocity about Y axis

$K_1$ : Ship speed gain

$n$  : Shaft angular speed

$\delta$  : Rudder angle

The block diagram of this system was shown in Appendix B.



## B. SIMULATION METHODS

### 1. Determine all initial conditions in the steady state

#### RUN 1.

Let propeller shaft speed command  $n^*$  be constant

$$n^* = 130 \text{ RPM}$$

And no feedback of ship speed

$$K1 = 0$$

Then determine the steady state speeds (i.e., determine the component of initial condition ship speed on X axis and on Y axis (UIC, VIC)).

#### RUN 2.

Now the propeller shaft speed command  $n^* = 130$  rpm making  $K1 = 8, 16, 24, 32$  respectively. Then determine the steady state in each case (i.e., find all IC's in the steady state). This guarantees that the loop is in steady state, since the initial conditions on other variables might still cause a transient.

2. Put in a step change in the ship speed with 2 kts. increase and delay time 10 sec. Then  $V^*$  becomes:

$$V^* = 20. + 2.0 \text{ step } (10.)$$

## C. SPEED CONTROL STUDIES

Using the initial conditions of RUN 1 and 2, a rudder command was applied at  $t = 0$ , causing a speed decrease due to added drag. At  $t = 10$  sec, the commanded speed,  $V^*$  was increased by 2 knots, stepwise. This caused a speed increase. The effects of changing the loop gain,  $K1$ , is shown in Figure E2.



#### D. DISCUSS THE CURVES

1. Ship resistance  $R_t$ , Wake fraction  $W$ , Thrust fraction  $t$

The ship speed change was small, so that:

- a. The ship resistance  $R_t = K.V$  (see Fig. B4)  $K = \text{constant}$

then the form of  $R_t$  was the same as  $V$  (Fig. E3 curve 1)

- b. The Wake fraction was constant (Fig. B1)

then  $W =$  (Fig. E3 curve 2)

- c. In this interval the slope of the thrust fraction was negative. Then when  $V$  decreases to increases and as  $V$  increases  $t$  decreases (Fig. E3 curve 3)

2.  $\sigma$  curve, torque coefficient  $C_q$ , thrusting coefficient

$$\begin{aligned}VP &= (1-W) V \\&= (1-.045) V \\&= .955 V\end{aligned}$$

$$\text{and } \sigma = \frac{nD}{V_p^2 - n^2 D^2}$$

The  $\sigma$  curve is shown in Fig. E4 curve 1

$\sigma$  changed from .853 to .920 then the slope of  $C_t$  and  $C_q$  curve was constant.

Then the  $C_t$  and  $C_q$  curve have the same form as the  $\sigma$  curve.

3. Propeller action torque  $Q_p$ , shaft torque  $Q_e$  and propeller angular speed  $n$

$$Q_p = C_q \cdot p \cdot D^3 (V_p^2 + n^2 D^2)$$

The  $Q_p$  curve was shown in Fig. E5 curve 2.

The  $W_f$  curve and  $n$  curve are shown in Figures E6 and E7.



	K1=0.	K1=8.	K1=16.	K1=24.	K1=32.
VIC		20.114	20.110	20.108	20.106
VIC	-0.10446	- 0.10446	- 0.10446	- 0.10446	- 0.10446
RIC		0.23219	0.23219	0.23219	0.23219
NIC		2.2056	2.2055	2.2055	2.2055
NRIC		-242.28	-454.09	-665.58	-876.55

TABLE 5 INITIAL CONDITIONS AT STEADY STATE

FOR K1=0,8,16,24,32





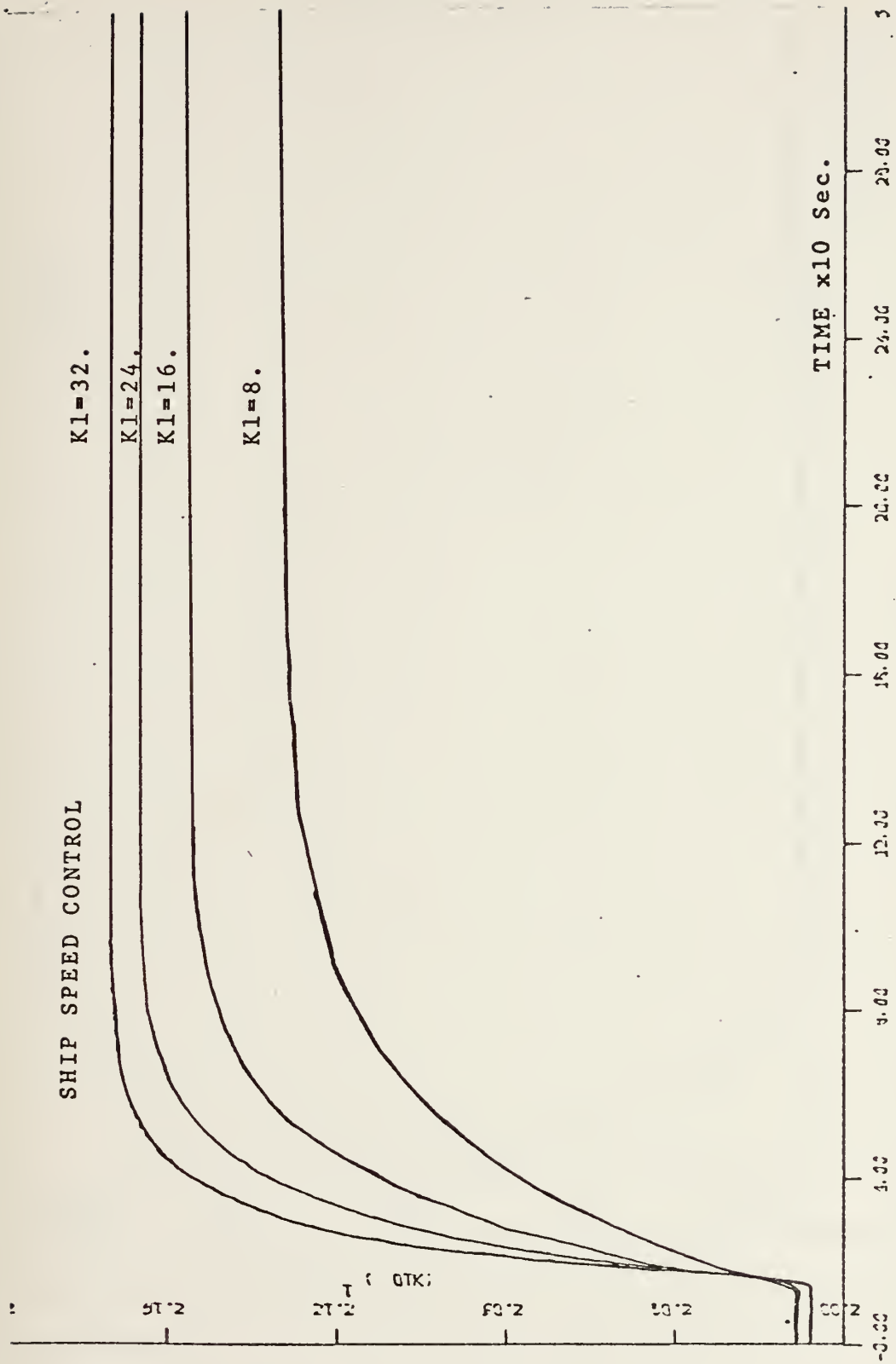


Figure E2. Ship Speed V versus Time, with  $K1 = 8, 16, 24, 32.$



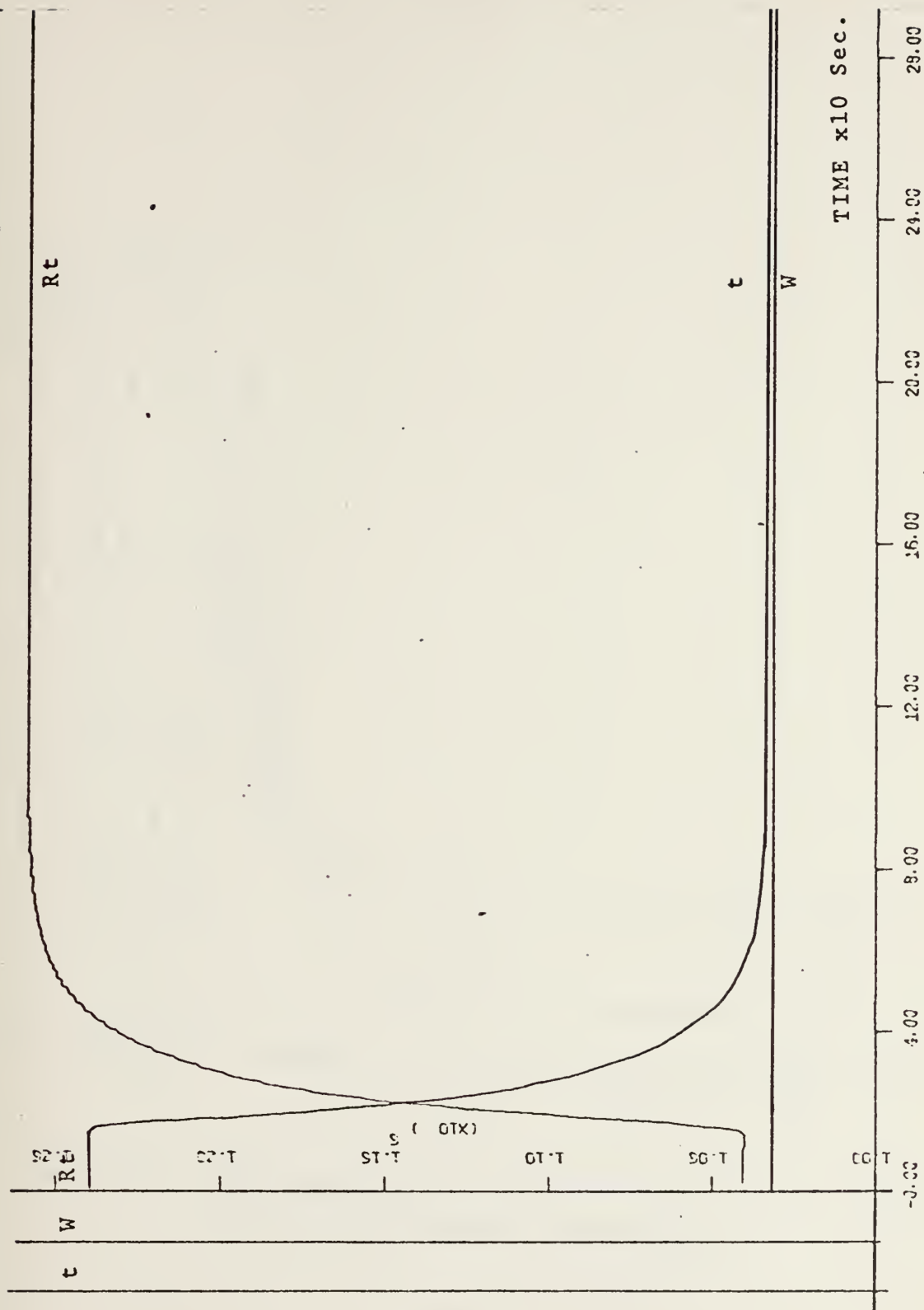


Figure E3. Ship resistance, Wake Fraction and Thrust Fraction versus Time.



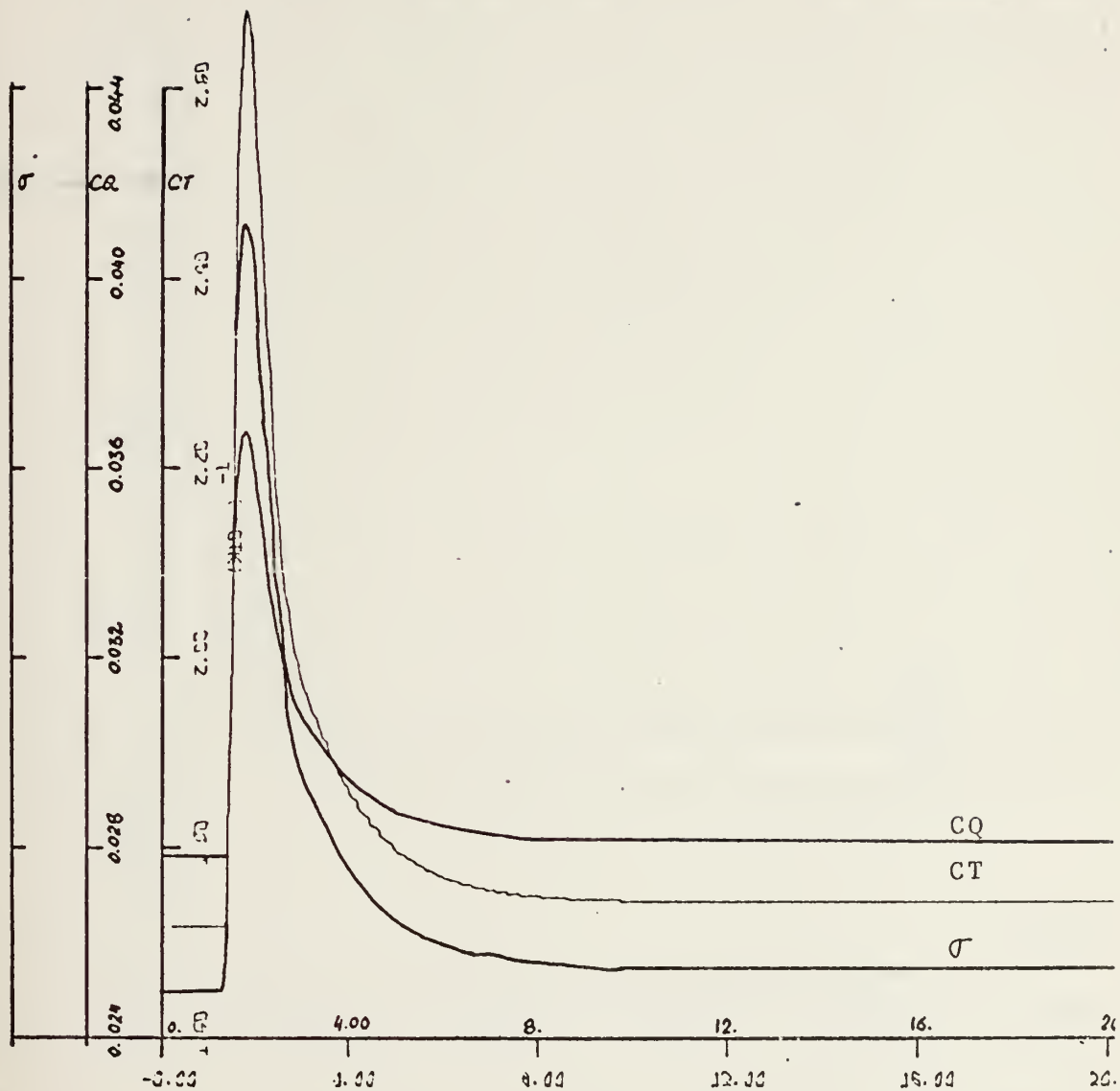


Figure E4. Second Modified Advance Coefficient  $\sigma$ , Torque Coefficient  $C_q$ , Thrust Coefficient  $C_t$  versus Time.



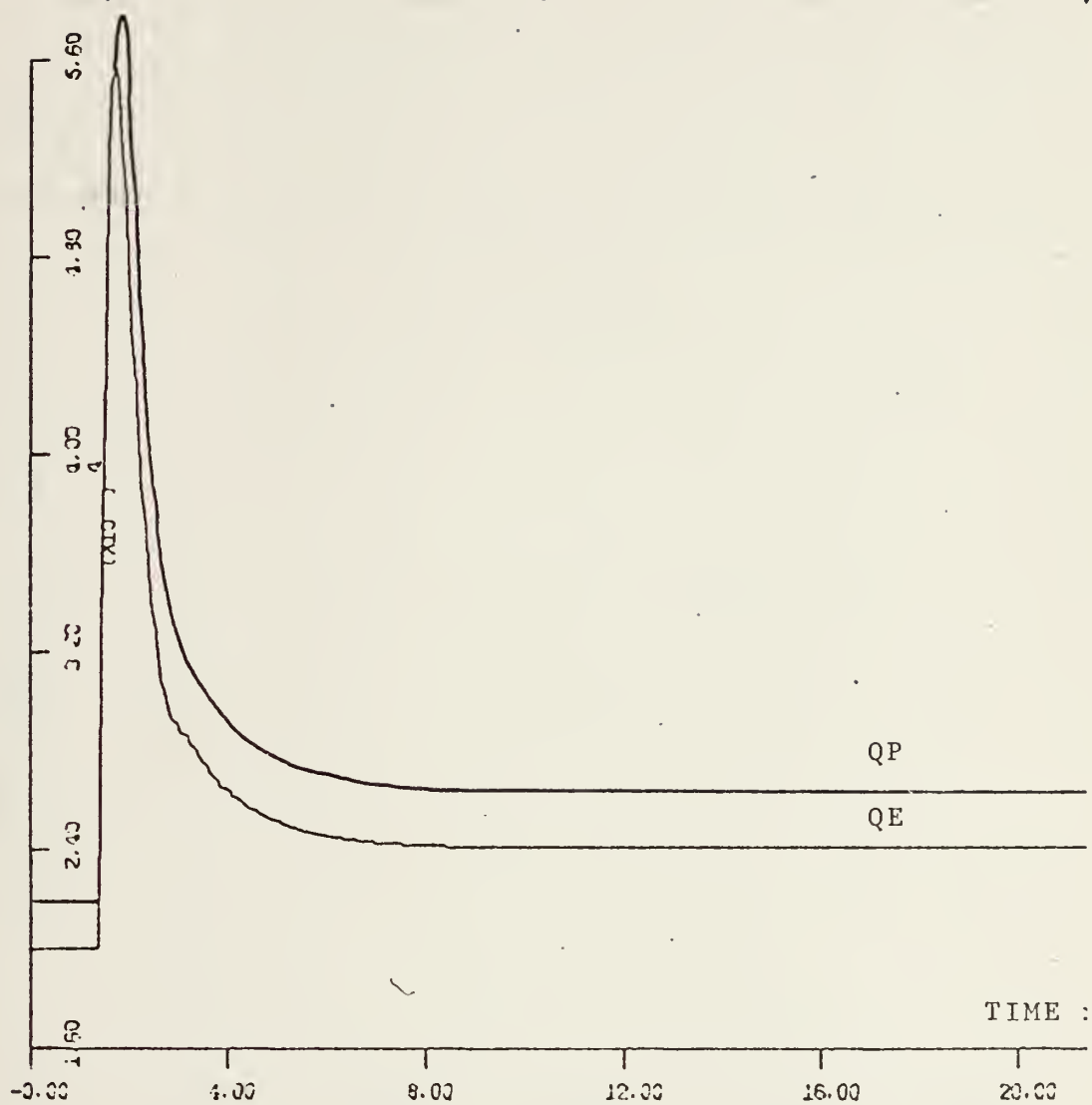


Figure E5. Propeller Action Torque  $Q_p$  and Shaft Torque versus Time.





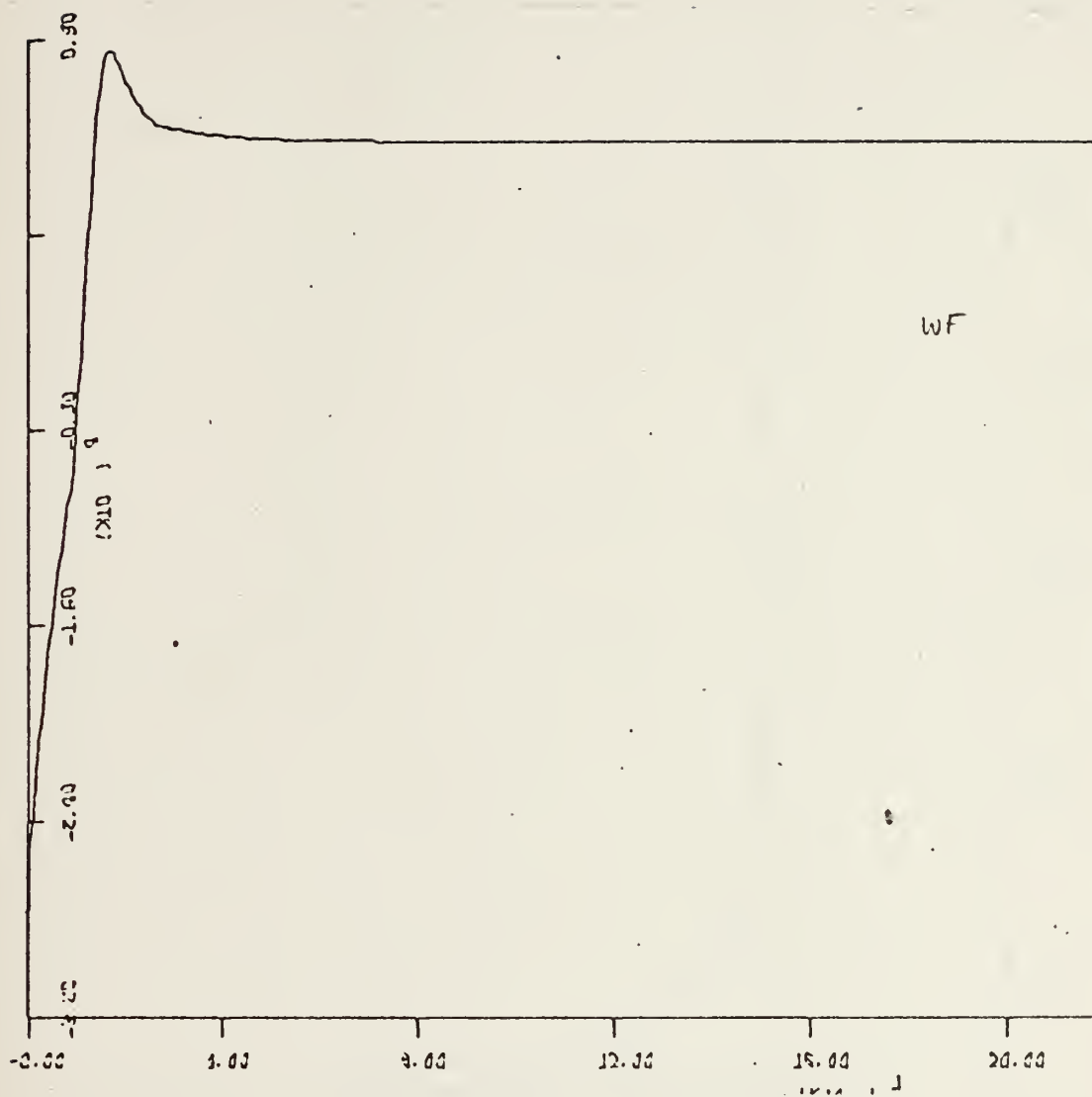


Figure E6. Fuel Flow Rate  $W_f$  versus Time.



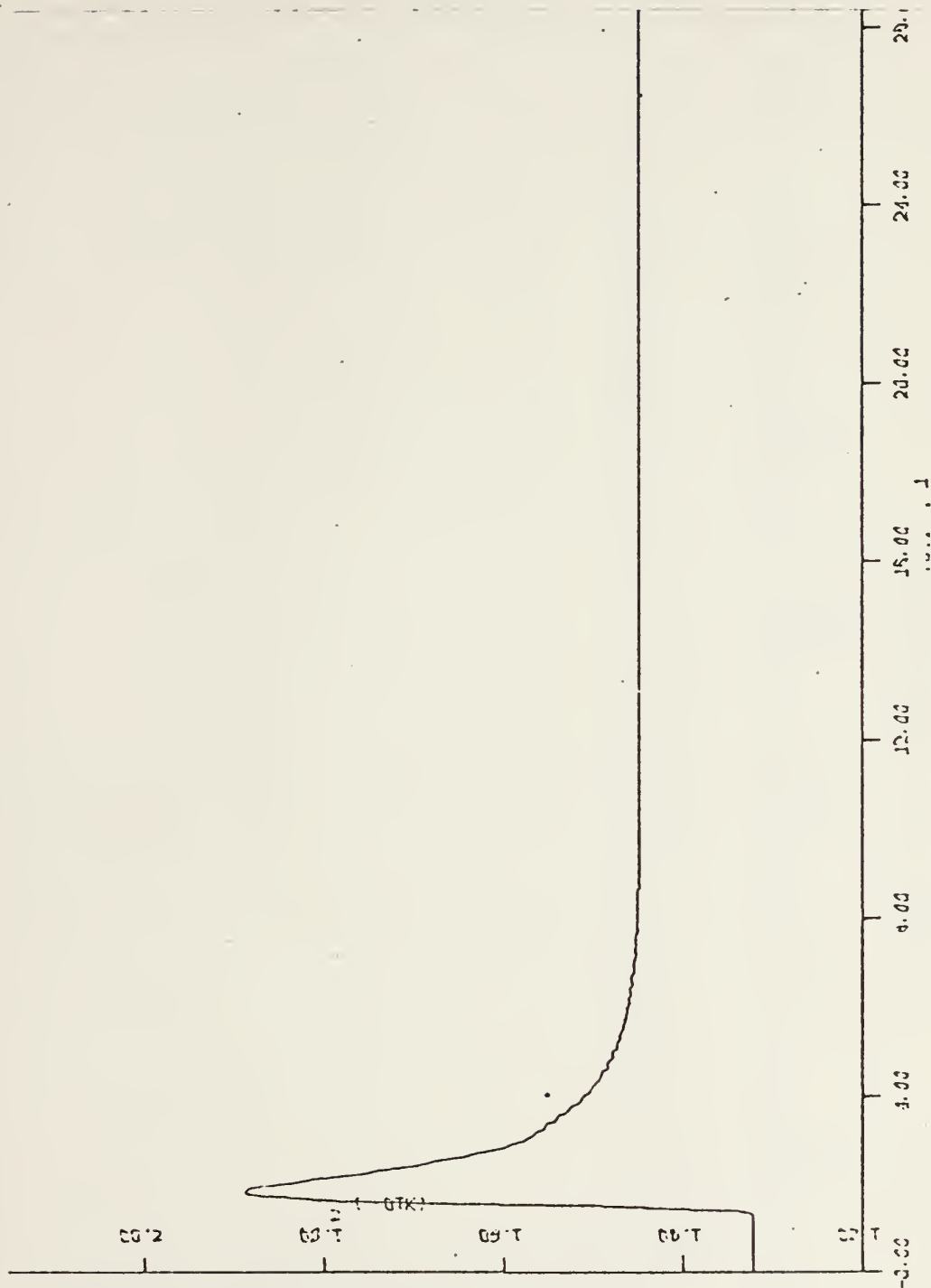


Figure E7. Propeller Angular Speed versus Time.





Figure E8. NER versus Time.



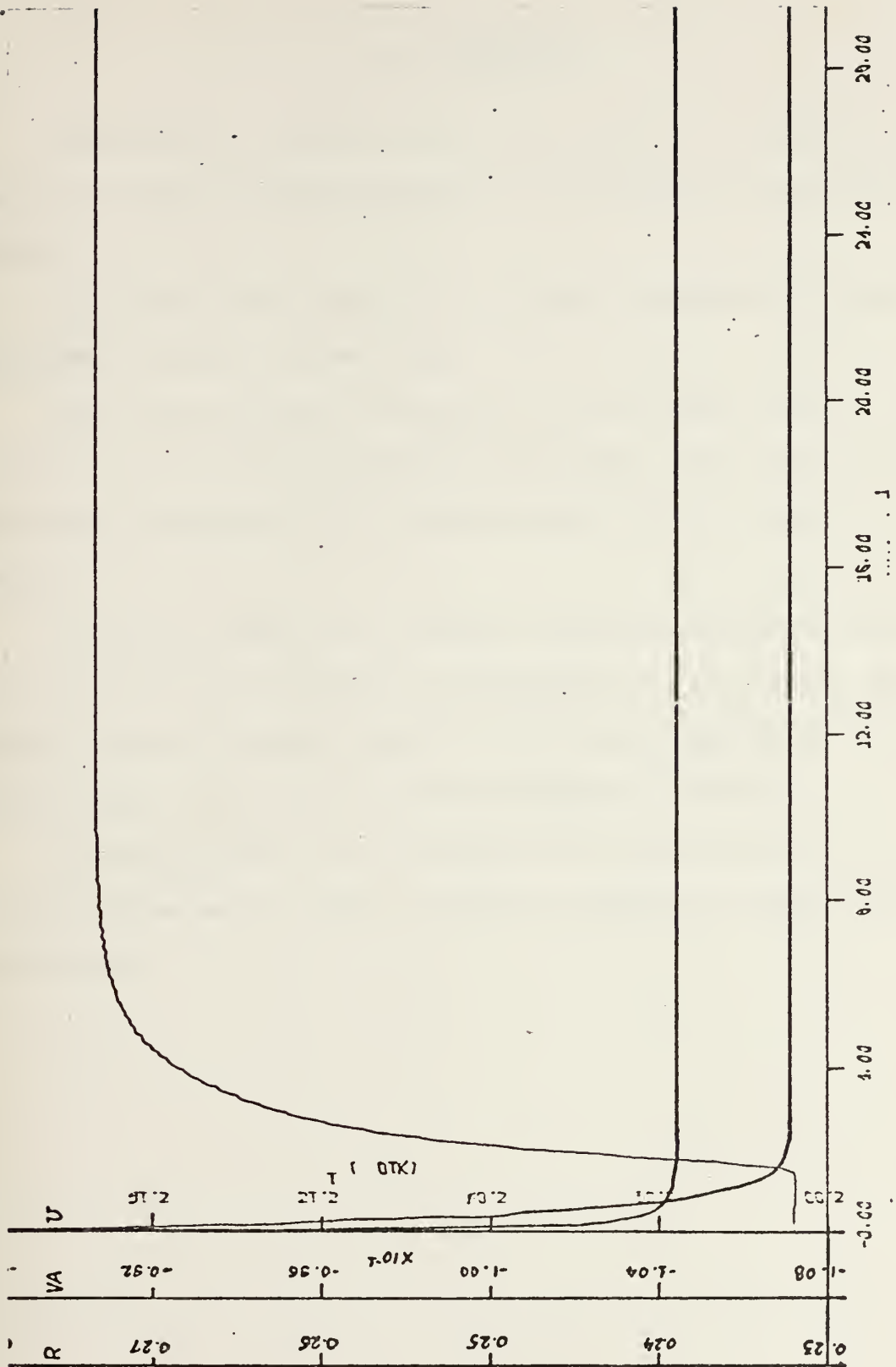


Figure E9. U, VA, R versus Time.





## VII. CONCLUSION

This thesis has presented the combined model of a propulsion plant and hull. DSL/360 LANGUAGE was found to be powerful for simulating ship motion.

In turbine powered ships, it is necessary to predict the propeller response to changes in power demand.

This problem has been investigated but the studies have been concentrated on the effect of dynamics of the ship. This work is concerned both with the dynamic of the propulsion plant and ship dynamics of the ship.

It has been shown that the complex, nonlinear dynamics of propulsion plant and hull can be adequately represented by an all digital computer model. The model permits studies of the internal dynamics of the propulsion plant, as well as the external dynamics of the hull.

It has been shown that the model can be incorporated in a feedback control system and its effects included in studies of control system performance.



## VIII. RECOMMENDATIONS

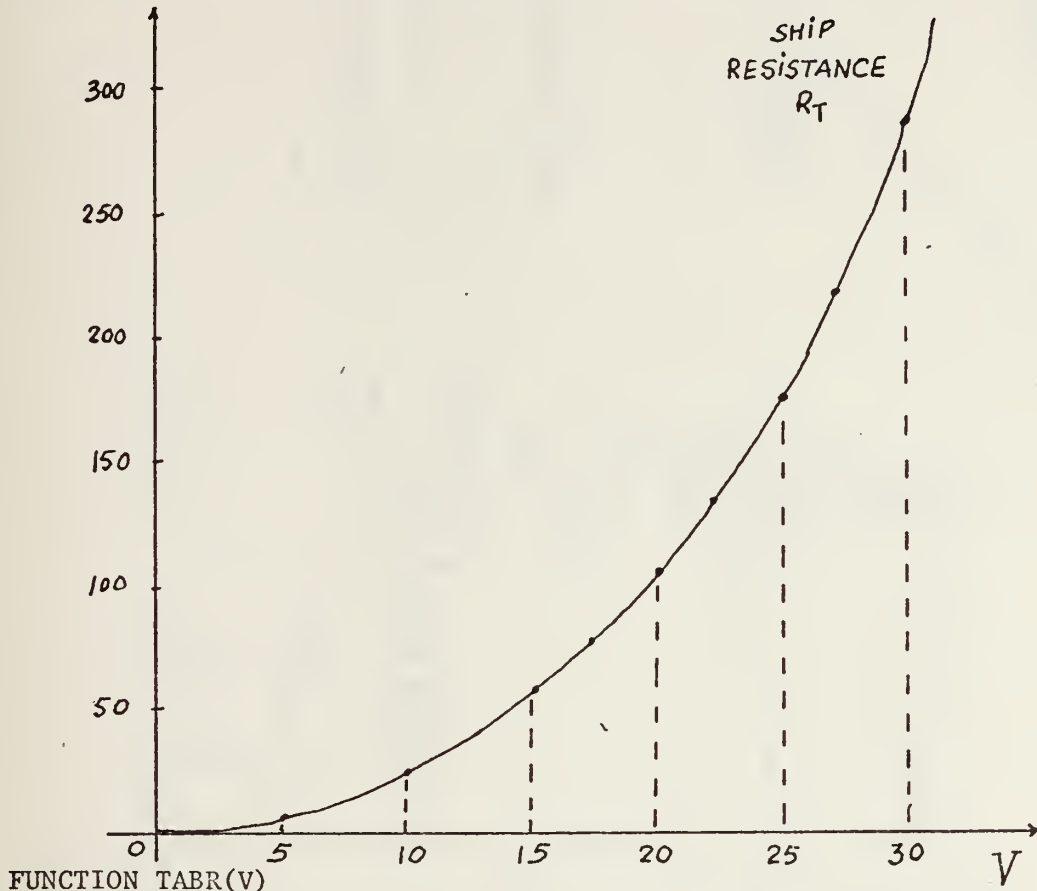
The combination of propulsion plant and hull in a single computer model provides a realistic tool for many studies. The following are topics suitable for future investigation.

- a. Course keeping, station keeping and replenishment at sea.
- b. Effect of propulsion plant dynamics on maneuvering in sea states (or regular waves).
- c. Effect of sea state conditions on the operation of the propulsion plant.
- d. Discuss variation in steady state speed with increase command as shown in Fig. C1 - C5. Also discuss effect of governor loop on steady state speed, Fig. E2.



# APPENDIX A

Table Look-up for Ship Resistance versus Speed:



FUNCTION TABR(V)

DIMENSION VT(13),RTT(13)

DATA VT/0.0,2.5,5.0,7.5,10.,12.5,15.,17.5,20.,22.5,25.5,27.5,30./

DATA RTT/2\*0.0,7000.,13000.,25000.,39000.,57000.,80000.,1103000.,  
136000.,173000.,225000.,280000./

IF(V.GT.30) GO TO 1

DELV 2.5

N IFIX(V/DELV) 1

SLOPR(RTT(N 1)-RTT(N))/(VT(N-1)-VT(N))

TABR SLOPR\*(V-VT(N))-RTT(N)

RETURN

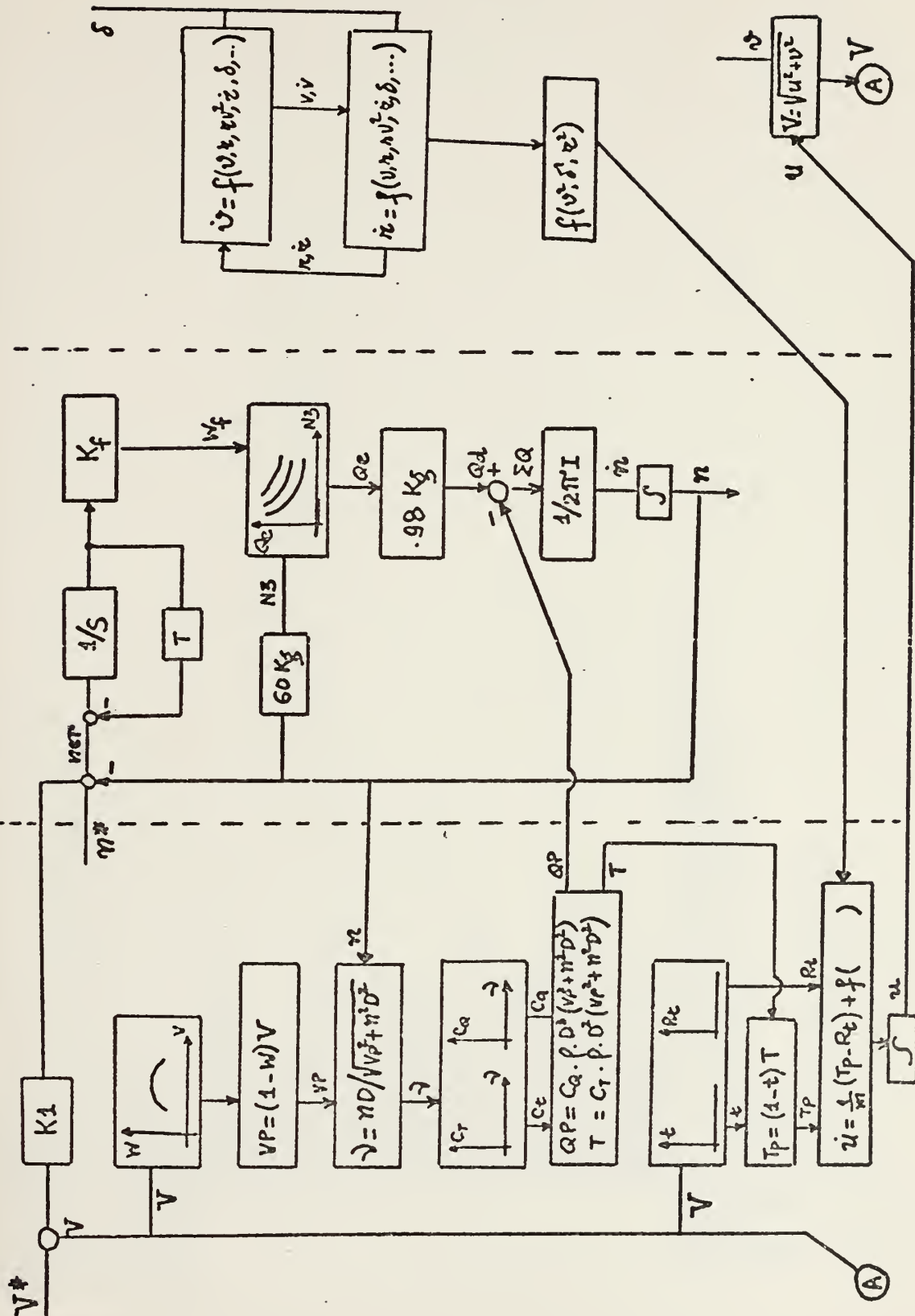
1 TABR 280000.

RETURN

END



# APPENDIX B







```

*      COMPUTER PROGRAM 1
// EXEC DSL
//DSL INPUT DD *
//INTEGR TRAPZ
//INTEGR NPLUT, NUM
CONST NPLUT=1
CONST KG=14., PI=3.1416, I=9.80E4, P=2.0
CONST D=15.
CONST M=1.292E+6
INCCCN NIC=2.0
INCCCN UIC=15.
DERIVATIVE
Y=STEP(0.0)
WF=7000.+3000.*Y
RT=TABR(V)
W=TABW(V)
T=TABT(V)
VP=(1.-W)*V
A=VP**2+(N*D)**2
B=(N*D)/B
CT=TABCT(S,VP)
TA=CT*P*D**2*A
TP=(1.-T)*TA
TPR=(TP-RT)/M
UDOT=TPR
U=INTGRL(UIC,UDOT)
V=U
CG=TABCG(S,VP)
GP=CQ*P*D**3*A
SUMQ=.98*KG*QE-QP
NDOT=SUMQ/(2.*PI*I)
N=INTGRL(NIC,NDOT)
N4=60.*N
N3=KG*N4
QE=TABQE(N3,WF)
SAMPLE 1.0,S,V,QP,QE,N4,SUMQ,CT,CQ,TPR
PRINT 1.0,QE,N3,QP,CT,CQ,N4,V,TPR
PREPAR FINT IM=160.,DELT=1.0,DELS=1.0
CONTRL TIME,V
GRAPH TIME,QE
PRPLCT CNLY
CALL DRWG(1,1,TIME,V)
CALL DRWG(2,1,TIME,QE)
TERMINAL
CALL ENDRW(NPLUT)
STOP

```



```

END
FORTRAN
FUNCTION TABW(V)
DIMENSION VT(13), WT(13)
DATA VT/0.0,0.2,5.5,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA WT/4*0.0,.005,.01,.02,.033,.045,.045,.038,.02,.004/
IF(V.GT.30.) GO TO 1
DELV=2.5
N=FIX(V/DELV)+1
SLOPW=(WT(N+1)-WT(N))/(VT(N+1)-VT(N))
TABW=SLOPW*(V-VT(N))+WT(N)
RETURN
TABW=0.0
1 RETURN
END
FUNCTION TABR(V)
DIMENSION VT(13), RTT(13)
DATA VT/0.0,0.2,5.5,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA RTT/2*0.0,7000.,13000.,25000.,39000.,57000.,80000.,103000.,
113600.,173000.,225000.,280000./
IF(V.GT.30.) GO TO 1
DELV=2.5
N=FIX(V/DELV)+1
SLOPR=(RTT(N+1)-RTT(N))/(VT(N+1)-VT(N))
TABR=SLOPR*(V-VT(N))+RTT(N)
RETURN
TABR=280000.
1 RETURN
END
FUNCTION TABT(V)
DIMENSION VT(13), TT(13)
DATA VT/0.0,0.2,5.5,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA TT/5*0.0,.01,.05,.07,.08,.075,.072,.06,.02/
IF(V.GT.30.) GO TO 1
DELV=2.5
N=FIX(V/DELV)+1
SLOPT=(TT(N+1)-TT(N))/(VT(N+1)-VT(N))
TABT=SLOPT*(V-VT(N))+TT(N)
RETURN
TABT=0.0
1 RETURN
END
FUNCTION TABCT(S,VP)
DIMENSION ST(21),CT1(21),CT2(21)
DATA ST/-1.,-.9,-.8,-.7,-.6,-.5,-.4,-.3,-.2,-.1,0.,.1,.2,.3,
1.4,1.5,1.6,1.7,1.8,.9,1./
DATA CT1/-.40,-.15,-.05,-.10,-.15,-.20,-.26,-.36,-.44,-.30,-.31,-.4,
1.45,-.42,-.40,-.39,-.37,-.32,-.33,-.50/

```



```

DATA CT1/-0.4,-0.2,-0.28,-0.32,-0.35,-0.38,-0.40,-0.41,-0.36,-0.29,-0.25,
1-0.35,-0.34,-0.25,-0.20,-0.13,-0.05,0.02,0.10,0.21,0.45/
DELS=0.1
N=IFIX((S+1.)/DELS)+1
IF(VP.LT.0.0) GO TO 2
SLOCCT1=(CT1(N+1)-CT1(N))/(ST(N+1)-ST(N))
TABCT=SLOCCT1*(S-ST(N))+CT1(N)
GO TO 6
2 SLOCCT2=(CT2(N+1)-CT2(N))/(ST(N+1)-ST(N))
TABCT=SLOCCT2*(S-ST(N))+CT2(N)
6 RETURN
END
FUNCTION TABCQ(S,VP)
DIMENSION ST(21),CQ1(21),CQ2(21)
DATA ST/-1,-0.9,-0.8,-0.7,-0.6,-0.5,-0.4,-0.3,-0.2,-0.1,0,.1,.2,.3,
1-0.4,0.5,0.6,0.7,0.8,0.9,1./
DATA CQ1/-0.08,-0.045,-0.05,-0.055,-0.06,-0.061,-0.062,-0.064,-0.058,-0.04,
1-0.037,-0.055,-0.05,-0.035,-0.028,-0.015,-0.008,0.005,0.02,0.032,0.07/
DATA CQ2/-0.08,-0.035,-0.015,0.0,0.01,0.02,0.034,0.042,0.06,0.07,0.045,0.05,
1-0.068,0.07,0.07,0.062,0.06,0.058,0.05,0.08/
DELS=0.1
N=IFIX((S+1.)/DELS)+1
IF(VP.LT.0.0) GO TO 2
SLOCQ1=(CQ1(N+1)-CQ1(N))/(ST(N+1)-ST(N))
TABCQ=SLOCQ1*(S-ST(N))+CQ1(N)
GO TO 6
2 SLOCQ2=(CQ2(N+1)-CQ2(N))/(ST(N+1)-ST(N))
TABCQ=SLOCQ2*(S-ST(N))+CQ2(N)
6 RETURN
END
FUNCTION TABQE(V3,WF)
DIMENSION V3T(10),WFT(10),QET(5,10)
V3T(1)=0.0
V3T(2)=1000.
V3T(3)=2000.
V3T(4)=3000.
V3T(5)=4000.
WFT(1)=2300.
WFT(2)=4300.
WFT(3)=5300.
WFT(4)=6300.
WFT(5)=7300.
WFT(6)=8300.
WFT(7)=9300.
WFT(8)=10300.
WFT(9)=11300.
WFT(10)=12300.
QET(1,1)=20000.

```









```

GET(5,10)=34000.
IF(V3.LT.0.) GO TO 1
IF(V3.GT.4000.) GO TO 2
X3=V3
GO TO 100
1 X3=0.0
GC TO 100
2 X3=4000.
GC TO 100
100 IF(WF.LT.3300.) GO TO 3
IF(WF.GT.12300.) GO TO 4
WF1=WF
GC TO 200
3 WF1=3300.
GC TO 200
4 WF1=12300.
J=FIX((WF1-3300.)/1000.)+1
DR=X3-V3T(I)
DW=WF1-WFT(J)
DELQW={DR/1000.}* (QET(I+1,J)-QET(I,J))
DELQW={DW/1000.}* (QET(I,J+1)-QET(I,J))
TABQW=QET(I,J)+DELQW+DELQW
RETURN
END
//PLCT.SYSIN DD *
TVAN N4

```

04

5.

8.

TVAN QE

04

5.

8.



COMPUTER PROGRAM 2

```

* // EXEC DSL DD *
// DSL INPUT DD *
// INTEGR TRAPZ
INCCCN NPLCT
INCCCN NIC=2.0
INCCCN NIC=15.0
INCCCN NRPLCT=2
INCCCN NRPLCT=120.
CONST K=14.,PI=3.1416,I=9.80E4,P=2.0
CONST M=1.292E+6
CONST KF=40.
CONST D=15.
CONST T=100.
CONSTRATIVE
Y=STEP(0.0)
NSTAR=120.+10.*Y
NER=NSTAR-N4
NDER=NER-T*NR
NK=INTGRL(NRIC,NDER)
DELF=KF*NR
WF=7000.+DELF
RT=TABR(V)
W=TABW(V)
T=TABT(V)
VP=(1.-W)*V
A=VP**2+(N*D)**2
B=SQRT(A)
S=(N*D)/B
CT=TABCT(S,VP)
TA=CT*P*D**2*A
TP=(1.0-T)*TA
TPR=(TP-RT)/M
UDCT=TPR
U=INTGRL(UIC,UDOT)
V=U
CQ=TABCQ(S,VP)
CP=CQ*P*D**3*A
SUMQ=.98*KG*QE-QP
NDCT=SUMQ/(2.*PI*I)
N=INTGRL(NIC,NDOT)
N4=60.*N
N3=KG*N4
QE=TABQE(N3,WF)

SAMPLE 1
PRINT 1.0,TPR,CQ,CT,N4,QE,QP,V
PREPAR 1.0,QE,N3,QP,CT,CQ,N4,V,TPR
CONTRL FINTIM=160.,DELT=0.5,DELS=0.5

```



```

GRAPH TIME,N4
GRAPH TIME,QE
PRPLOT CNLY
CALL DRWG(1,1,TIME,QE)
CALL DRWG(2,1,TIME,N4)
TERMINAL
CALL ENDRW(NPLOT)
STOP
END
FORTRAN
FUNCTION TABT(V)
DIMENSION VT(13),TT(13)
DATA VT/0.0,2.5,5.5,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA TT/5*0.0,0.01,0.05,0.07,0.08,0.075,0.072,0.06,0.02/
DELV=2.5
N=FIX(V/DELV)+1
SLOPT=(TT(N+1)-TT(N))/(VT(N+1)-VT(N))
TABT=SLOPT*(V-VT(N))+TT(N)
RETURN
END
FUNCTION TABR(V)
DIMENSION VT(13),RTT(13)
DATA VT/0.0,2.5,5.5,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA RTT/2*0.0,7000.,13000.,25000.,39000.,57000.,80000.,103000.,
1136000.,173000.,225000.,280000./
DELV=2.5
N=FIX(V/DELV)+1
SLOPR=(RTT(N+1)-RTT(N))/(VT(N+1)-VT(N))
TABR=SLOPR*(V-VT(N))+RTT(N)
RETURN
END
FUNCTION TABW(V)
DIMENSION VT(13),WT(13)
DATA VT/0.0,2.5,5.5,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA WT/4*0.0,0.005,0.01,0.02,0.033,0.045,0.045,0.02,0.004/
DELV=2.5
N=FIX(V/DELV)+1
SLOPW=(WT(N+1)-WT(N))/(VT(N+1)-VT(N))
TABW=SLOPW*(V-VT(N))+WT(N)
RETURN
END
FUNCTION TABCQ(S,VP)
DIMENSION ST(21),CQ1(21),CQ2(21)
DATA ST/-1.,-9,-8,-7,-6,-5,-4,-3,-2,-1,0.,.1,.2,.3,
1.4,2.5,6.,7.,8.,9,1./
DATA CQ1/-0.8,-.045,-.05,-.06,-.061,-.062,-.064,-.058,-.04,
1-.037,-.055,-.05,-.035,-.015,0.0,0.01,0.02,0.034,0.042,0.06,0.07,0.045,0.05,
DATA CQ2/-0.08,-.035,-.015,0.0,0.01,0.02,0.034,0.042,0.06,0.07,0.045,0.05,

```



```

1 068,.07,.07,.062,.06,.058,.058,.05,.08/
DELS=0.1
N=IFIX((S+1.)/DELS)+1
IF(VP.LT.0.0) GO TO 2
SLOCQ1=(CQ1(N+1)-CQ1(N))/(ST(N+1)-ST(N))
TABCQ=SLOCQ1*(S-ST(N))+CQ1(N)
GC TO 6
SLOCQ2=(CQ2(N+1)-CQ2(N))/(ST(N+1)-ST(N))
TABCQ=SLOCQ2*(S-ST(N))+CQ2(N)
RETURN
END
FUNCTION TABCT(S,VP)
DIMENSION ST(21),CT1(21),CT2(21)
DATA ST/-1.7,-.9,-.8,-.7,-.6,-.5,-.4,-.3,-.2,-.1,0,.1,.2,.3,
1.4,.5,.6,.7,.8,.9,1./
DATA CT2/-40,-.40,-.15,-.05,.10,.15,.20,.26,.36,.44,.30,.31,.4,
1.45,.42,.40,.39,.37,.32,.33,.50/
DATA CT1/-4,-.4,-.2,-.28,-.32,-.35,-.38,-.40,-.41,-.36,-.29,-.25,
1-.35,-.34,-.25,-.20,-.13,-.05,.02,.10,.21,.45/
DELS=0.1
N=IFIX((S+1.)/DELS)+1
IF(VP.LT.0.0) GO TO 2
SLOCCT1=(CT1(N+1)-CT1(N))/(ST(N+1)-ST(N))
TABCT=SLOCCT1*(S-ST(N))+CT1(N)
GC TO 6
SLOCCT2=(CT2(N+1)-CT2(N))/(ST(N+1)-ST(N))
TABCT=SLOCCT2*(S-ST(N))+CT2(N)
RETURN
END
FUNCTION TABQE(V3,WF)
DIMENSION V3T(5),WFT(10),QET(5,10)
V3T(1)=0.0
V3T(2)=1000.
V3T(3)=2000.
V3T(4)=3000.
V3T(5)=4000.
WFT(1)=3000.
WFT(2)=4300.
WFT(3)=5300.
WFT(4)=6300.
WFT(5)=7300.
WFT(6)=8300.
WFT(7)=9300.
WFT(8)=10300.
WFT(9)=11300.
WFT(10)=12300.
QET(1,1)=20000.
QET(1,2)=28500.

```





QETI(1,1,3)=37000.  
QETI(1,1,4)=45000.  
QETI(1,1,5)=53500.  
QETI(1,1,6)=62000.  
QETI(1,1,7)=70000.  
QETI(1,1,8)=77000.  
QETI(1,1,9)=86500.  
QETI(1,1,10)=95000.  
QETI(1,2,1)=12500.  
QETI(1,2,2)=22000.  
QETI(1,2,3)=27000.  
QETI(1,2,4)=34500.  
QETI(1,2,5)=41500.  
QETI(1,2,6)=48000.  
QETI(1,2,7)=54000.  
QETI(1,2,8)=59000.  
QETI(1,2,9)=64000.  
QETI(1,2,10)=69000.  
QETI(1,3,1)=8000.  
QETI(1,3,2)=13000.  
QETI(1,3,3)=18500.  
QETI(1,3,4)=24000.  
QETI(1,3,5)=29000.  
QETI(1,3,6)=34000.  
QETI(1,3,7)=39500.  
QETI(1,3,8)=45000.  
QETI(1,3,9)=50000.  
QETI(1,3,10)=55000.  
QETI(1,4,1)=5000.  
QETI(1,4,2)=9000.  
QETI(1,4,3)=13000.  
QETI(1,4,4)=17500.  
QETI(1,4,5)=22000.  
QETI(1,4,6)=26000.  
QETI(1,4,7)=30000.  
QETI(1,4,8)=34000.  
QETI(1,4,9)=38500.  
QETI(1,4,10)=43000.  
QETI(1,5,1)=3000.  
QETI(1,5,2)=6500.  
QETI(1,5,3)=10000.  
QETI(1,5,4)=13000.  
QETI(1,5,5)=17000.  
QETI(1,5,6)=20000.  
QETI(1,5,7)=24000.  
QETI(1,5,8)=27000.  
QETI(1,5,9)=30500.  
QETI(1,5,10)=34000.  
QETI(1,1,1)=37000.  
QETI(1,1,2)=45000.  
QETI(1,1,3)=53500.  
QETI(1,1,4)=62000.  
QETI(1,1,5)=70000.  
QETI(1,1,6)=77000.  
QETI(1,1,7)=86500.  
QETI(1,1,8)=95000.  
QETI(1,2,1)=12500.  
QETI(1,2,2)=22000.  
QETI(1,2,3)=27000.  
QETI(1,2,4)=34500.  
QETI(1,2,5)=41500.  
QETI(1,2,6)=48000.  
QETI(1,2,7)=54000.  
QETI(1,2,8)=59000.  
QETI(1,2,9)=64000.  
QETI(1,2,10)=69000.  
QETI(1,3,1)=8000.  
QETI(1,3,2)=13000.  
QETI(1,3,3)=18500.  
QETI(1,3,4)=24000.  
QETI(1,3,5)=29000.  
QETI(1,3,6)=34000.  
QETI(1,3,7)=39500.  
QETI(1,3,8)=45000.  
QETI(1,3,9)=50000.  
QETI(1,3,10)=55000.  
QETI(1,4,1)=5000.  
QETI(1,4,2)=9000.  
QETI(1,4,3)=13000.  
QETI(1,4,4)=17500.  
QETI(1,4,5)=22000.  
QETI(1,4,6)=26000.  
QETI(1,4,7)=30000.  
QETI(1,4,8)=34000.  
QETI(1,4,9)=38500.  
QETI(1,4,10)=43000.  
QETI(1,5,1)=3000.  
QETI(1,5,2)=6500.  
QETI(1,5,3)=10000.  
QETI(1,5,4)=13000.  
QETI(1,5,5)=17000.  
QETI(1,5,6)=20000.  
QETI(1,5,7)=24000.  
QETI(1,5,8)=27000.  
QETI(1,5,9)=30500.  
QETI(1,5,10)=34000.



```

IF(WF.LE.3300.) WF=3300.
IF(WF.GE.12100.) WF=12100.
IF(V3.LE.0.0) V3=0.0
IF(V3.GE.4000.) V3=4000.
I=IFIX((V3/1000.))+1
J=IFIX((WF-3300.)/1000.)+1
IF(I.GT.5) I=5
IF(J.GT.10) J=10
IF((I.EQ.5).OR.(J.EQ.10)) GO TO 1
DR=V3-V3T(I)
DW=WF-WFT(J)
DELQR=((DR/1000.)*(QET(I+1,J)-QET(I,J))
DELQW=((DW/1000.)*(QET(I,J+1)-QET(I,J))
TARQE=QET(I,J)+DELQR+DELQW
RETURN
TABQE=QET(I,J)
RETURN
END

```

1

```

//PLCT·SYSIN DD *
TVAN QE

```

TVAN N42

0.0

20.

120.

20.

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05



```

*//TVAN3      JOB      COMPUTER PROGRAM      3
//EXEC      DSL      (1345,0536,NE24), 'TVAN', TIME=1
//DSL      INPUT      DD *
INTEGER      NPLCT, NUM
CONST      NPLCT=1
CONST      TDMAX=20., DRATE=40.
CONST      N5=-101, N6=-1.42, N7=-.024, N8=.0202, N9=0.0
CONST      N1=-.0003, N2=-.075, N3=-.385, N4=-.306, N5=-.0569
CONST      N6=-.0461, Y6=-.0994, Y7=.05, Y8=0.0, Y9=.309
CONST      Y5=.00016, Y1=-.260, Y2=-2.15, Y3=-1.18, Y4=-.0781
INCCN      XIC=0.0, VIC=0.0, RIC=0.0
INCCN      UIC=15.
INCCN      PHIC=0.0
INITIAL
DERIVATIVE
DETA=Y9*N8-N5*Y8
D=-DRATE*(RAMP(0.0)-RAMP(TDMAX/DRATE))/57.3
U=INTGRL(UIC, UDOT)
VA=INTGRL(VIC, VDOT)
R=INTGRL(RIC, RDOT)
PHI=INTGRL(PHIC, R)
F1C=Y0+Y1*VA+Y2*VA**3+Y3*VA**R*Y4*Y5**R**3+Y6**R*VA*Y7*D
F2C=Y0+N1*VA+N2*VA**3+N3*VA**R*Y4*Y5**R**3+N6**R*VA*Y7*D
NUMV=N8*F1C-Y8*F2C
NUMR=Y9*F2C-N9*F1C
VDOT=NUMV/DETA
RDOT=NUMR/DETA
XCCT=U*COS(PHI)-VA*SIN(PHI)
YDOT=U*SIN(PHI)+VA*COS(PHI)
X=INTGRL(XIC, XDOT)
Y=INTGRL(YIC, YDOT)
SAMPLE      .2, X, Y, PHI, VA, R, VDOT, RDOT
PRINT      0.2, X, Y, PHI, VDOT, RDOT, VA, R
PREPAR      FINTIM=39., DELT=0.01, DELS=0.02
CUNTRL      CALL DRWG(1,1,X,Y)
TERMINAL
CALL      ENDRW(NPLCT)
END
STOP
//PLCT.SYSIN DD *
TVAN U=1.0 2.1 20 DEGREE OF RUDDER ANGLES 2.1 5.
-4.

```









```

U=INTGRL(UIC,UDGT)
CC=TABCQ(S,VP)
GF=CC*P*DA**3*A
SUMQ=.98*KG*QE-QP
N=INTGRL(NIC,NDOT)
N4C=60.*N
N3C=KG*N4C
QE=TABQE(N3,WF)
D=-DRATE*(RAMP(0.0)-RAMP(TDMAX/DRATE))/57.3
VA=INTGRL(VIC,VDOOT)
R=INTGRL(RIC,RDOOT)
PHI=INTGRL(PHIC,R)
F2C=N0+N1*VA+N2*VA**3+N3*VA**R*R+N4**R+N5**R**3+N6**R*VA+N7**D
F1C=Y0+Y1*VA+Y2*VA**3+Y3*VA**R*R+Y4**R+Y5**R**3+Y6**R*VA+Y7**D
NUMV=N8*F1C-N9*F2C
NUMR=N9*F2C-N9*F1C
VDOOT=NUMV/DETA
RDOOT=NUMR/DETA
XDOT=U*COS(PHI)-VA*SIN(PHI)
YDOT=U*SIN(PHI)+VA*COS(PHI)
Y=INTGRL(YIC,YDOOT)
X=INTGRL(XIC,XDOOT)
SQ=U**2+VA**2
V=SQRT(SQ)
YAL=STEP(10.0)
VSTAR=20.+2.*YAL
VER=VSTAR-V
KVER=K1*VER
SAMPLE FINTIM=320.,DELT=0.5,DELS=0.5
CTRL O.5,QE,N4,WF,CQ,CT,QP,S,T,WT,RT
PREPAR CALL DRWG(1,1,TIME,WF)
CALL DRWG(2,1,TIME,QE)
CALL DRWG(3,1,TIME,N4)
CALL DRWG(4,1,TIME,CQ)
CALL DRWG(5,1,TIME,CT)
CALL DRWG(6,1,TIME,QP)
CALL DRWG(7,1,TIME,S)
TERMINAL CALL ENDRW(NPLOT)
STOP
END
FCRTRAN FUNCTION TABW(V)
DIMENSION VT(13),WT(13)
DATA VT/0.0,2.5,5.,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA WT/4*0.0;.005;.01;.02;.033;.045;.045;.02;.004/

```



```

IF(V.GT.30.) GO TO 1
DELV=2.5
N=FIX(V/DELV)+1
SLOPW=(WT(N+1)-WT(N))/(VT(N+1)-VT(N))
TABW=SLOPW*(V-VT(N))+WT(N)
RETURN
TABW=0.0
RETURN
END
FUNCTION TABR(V)
DIMENSION VT(13), RTT(13)
DATA VT/0.0,2.5,5.,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA RTT/2*0.0,7000.,13000.,25000.,39000.,57000.,80000.,103000.,
1136000.,173000.,225000.,280000./
IF(V.GT.30.) GO TO 1
DELV=2.5
N=FIX(V/DELV)+1
SLOPR=(RTT(N+1)-RTT(N))/(VT(N+1)-VT(N))
TABR=SLOPR*(V-VT(N))+RTT(N)
RETURN
TABR=280000.
RETURN
END
FUNCTION TABT(V)
DIMENSION VT(13), TT(13)
DATA VT/0.0,2.5,5.,7.5,10.,12.5,15.,17.5,20.,22.5,25.,27.5,30./
DATA TT/5*0.0,0.01,0.05,0.07,0.08,0.075,0.072,0.06,0.02/
IF(V.GT.30.) GO TO 1
DELV=2.5
N=FIX(V/DELV)+1
SLOPT=(TT(N+1)-TT(N))/(VT(N+1)-VT(N))
TABT=SLOPT*(V-VT(N))+TT(N)
RETURN
TABT=0.0
RETURN
END
FUNCTION TABCT(S,VP)
DIMENSION ST(21), CT1(21), CT2(21)
DATA ST/-1.,-.9,-.8,-.7,-.6,-.5,-.4,-.3,-.2,-.1,0.,.1,0.2,0.3,
1.4,1.5,1.6,1.7,1.8,1.9,1./
DATA CT2/-0.4,0.15,-0.05,0.05,10.,15.,20.,26.,36.,44.,30.,31.,.4,
1.45,0.42,0.39,0.37,0.32,0.33,0.50/
DATA CT1/-0.4,0.2,-0.28,-0.32,-0.35,-0.38,-0.40,-0.41,-0.36,-0.29,-0.25,
1-0.35,-0.34,-0.25,-0.20,-0.13,-0.05,0.02,0.10,0.21,0.45/
DELS=0.1
N=FIX((S+1.)/DELS)+1
IF(VP.LT.0.0) GO TO 2
SLOCT1=(CT1(N+1)-CT1(N))/(ST(N+1)-ST(N))

```



```

2 TABCT=SLOCT1*(S-ST(N))+CT1(N)
  GO TO 6
  SLOCT2=(CT2(N+1)-CT2(N))/(ST(N+1)-ST(N))
  TABCT=SLOCT2*(S-ST(N))+CT2(N)
  RETURN
6 END
  FUNCTION TABCQ(S,VP)
  DIMENSION ST(21),CQ1(21),CQ2(21)
  DATA ST/-1.,-.9,-.8,-.7,-.6,-.5,-.4,-.3,-.2,-.1,0.,.1,.2,.3,
1.4,.5,.6,.7,.8,.9,1./
  DATA CQ1/-0.8,-.045,-.05,-.06,-.061,-.062,-.064,-.058,-.04,
1-.037,-.055,-.05,-.035,-.028,-.015,-.008,-.005,-.02,-.032,-.07/
  DATA CQ2/-0.8,-.035,-.015,0.0,-.01,-.02,-.034,-.042,-.06,-.07,-.045,-.05,
1.068,-.07,-.07,-.062,-.06,-.058,-.05,-.08/
  DELS=0.1
  N=IFIX((S+1.)/DELS)+1
  IF(VP.LT.0.0) GO TO 2
  SLOCT1=(CQ1(N+1)-CQ1(N))/(ST(N+1)-ST(N))
  TABCQ=SLOCT1*(S-ST(N))+CQ1(N)
  GO TO 6
  SLOCT2=(CQ2(N+1)-CQ2(N))/(ST(N+1)-ST(N))
  TABCQ=SLOCT2*(S-ST(N))+CQ2(N)
  RETURN
2 END
6 FUNCTION TABQE(V3,WF)
  DIMENSION V3T(5),WFT(10),QET(5,10)
  V3T(1)=0.0
  V3T(2)=1000.
  V3T(3)=2000.
  V3T(4)=3000.
  V3T(5)=4000.
  WFT(1)=3300.
  WFT(2)=4300.
  WFT(3)=5300.
  WFT(4)=6300.
  WFT(5)=7300.
  WFT(6)=8300.
  WFT(7)=9300.
  WFT(8)=10300.
  WFT(9)=11300.
  WFT(10)=12300.
  QET(1,1)=20000.
  QET(1,2)=28500.
  QET(1,3)=37000.
  QET(1,4)=45000.
  QET(1,5)=53500.
  QET(1,6)=62000.
  QET(1,7)=70000.

```



TO GO TO 2





```

2      GC TO 100
      X3=4000.
100    GC TO 100
      IF(WF.LT.3300.) GO TO 3
      IF(WF.GT.12300.) GO TO 4
      WF1=WF
      GC TO 200
      WF1=3300.
      GC TO 200
      WF1=12300.
200    I=FIX(X3/1000.)+1
      J=FIX((WF1-3300.)/1000.)+1
      DR=X3-V3T(I)
      DW=WF1-WF(J)
      DELQR=(DR/1000.)*((QET(I+1,J))-QET(I,J))
      DELQW=(DW/1000.)*((QET(I,J+1))-QET(I,J))
      TABQE=QET(I,J)+DELQR+DELQW
      RETURN
      END

```

```

//PLCT.SYSIN DD *
TVAN WF

```

TVAN QE	8.	5.	04
TVAN N4	8.	5.	04
TVAN CG	8.	5.	04
TVAN CT	8.	5.	04
TVAN QP	8.	5.	04
TVAN S	8.	5.	04

```

PREPAR 0.5,V,U
      CALL DRWG(1,1,TIME,V)
CONST K1=32.
INCCN NRIC=-862.84
INCCN NIC=2.2049

```







## LIST OF REFERENCES

1. PRINCIPLES OF NAVAL ARCHITECTURE, SNAME, 1968.
2. Thaler and Brown, Analysis and Design of Feedback Control System, McGraw-Hill, 1960.
3. M. A. Abkowitz, Stability and Motion Control of Ocean Vehicles, MIT, 1969.
4. Proceedings Ship Control Systems, Symposium, 2nd, 1969.
5. DSL/360 User Manual, IBM, 1968.



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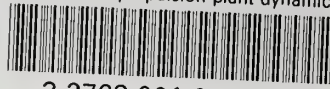
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